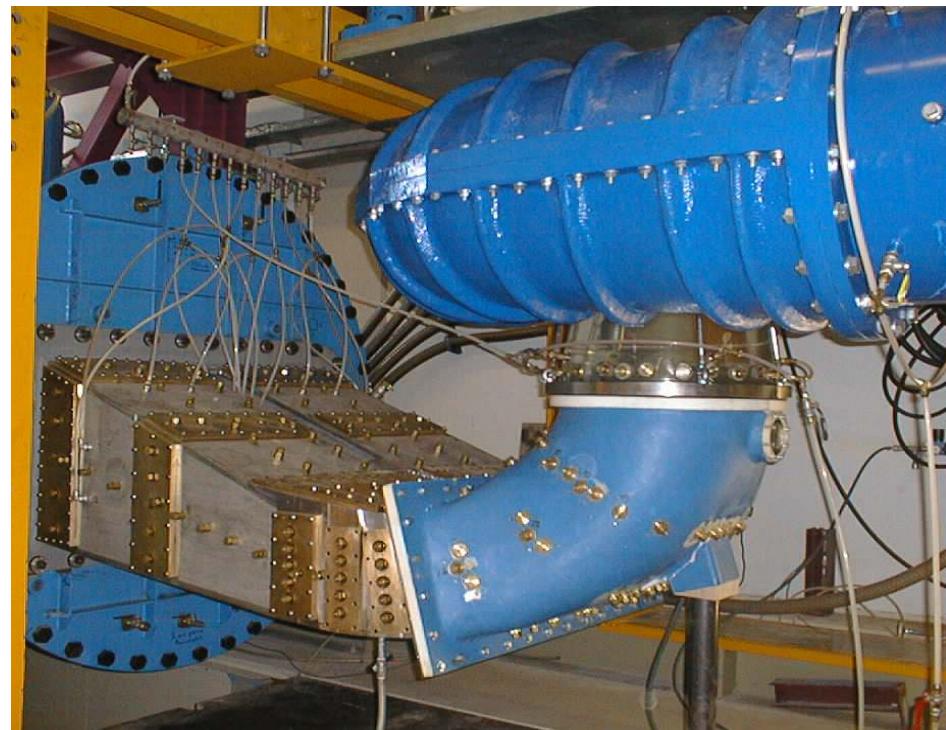


## **Metode moderne pentru cercetarea in Masini Hidraulice**



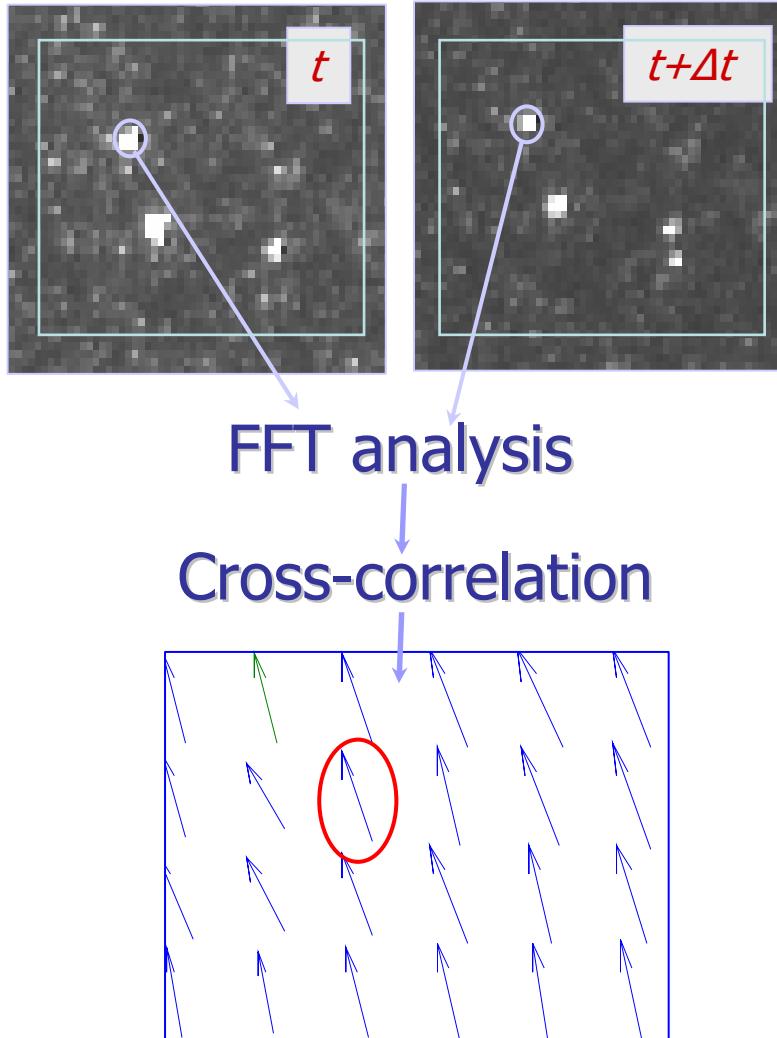
*Dr. Gabriel Dan CIOCAN  
Associate Professor*



# Contents

## New Experimental Methods and New Theoretical Developments:

## 3D PIV Principle



✓ *Acquisition Parameters*

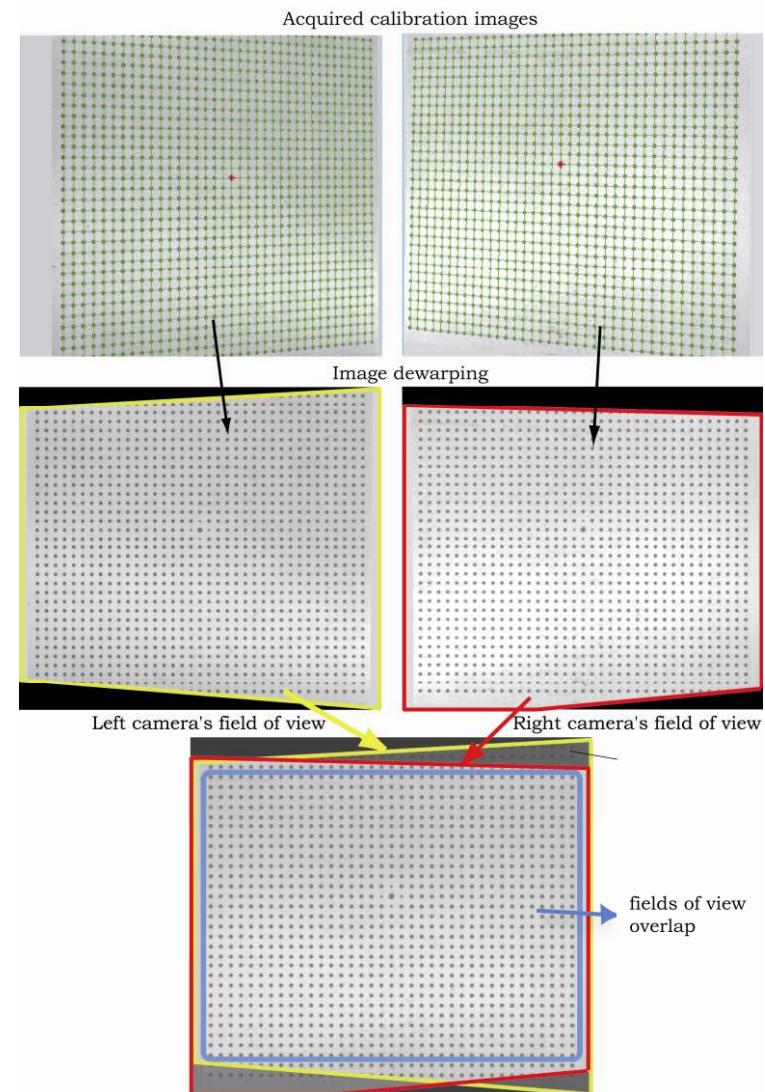
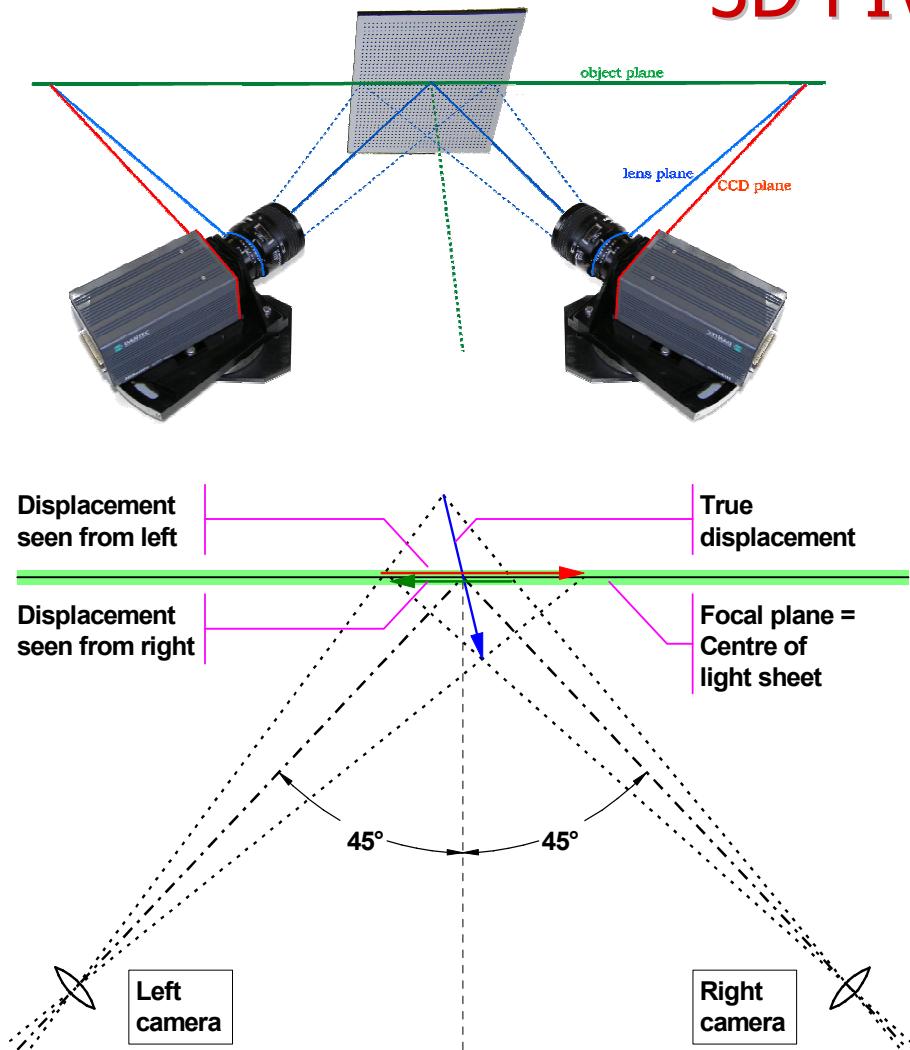
- time interval
- seeding density
- energy distribution in the laser-sheet
- laser-sheet alignment

✓ *Vectors Fields Validation*

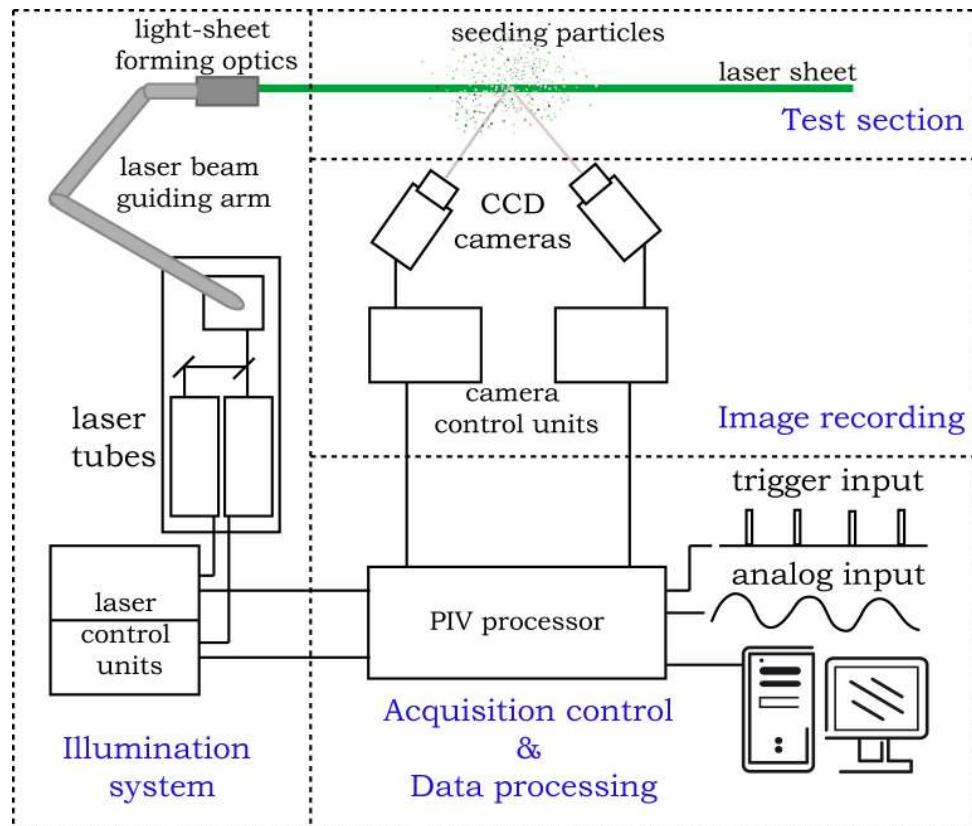
- masking
- correlation quality
- moving average interpolation

# Metode moderne pentru cercetarea in Masini Hidraulice

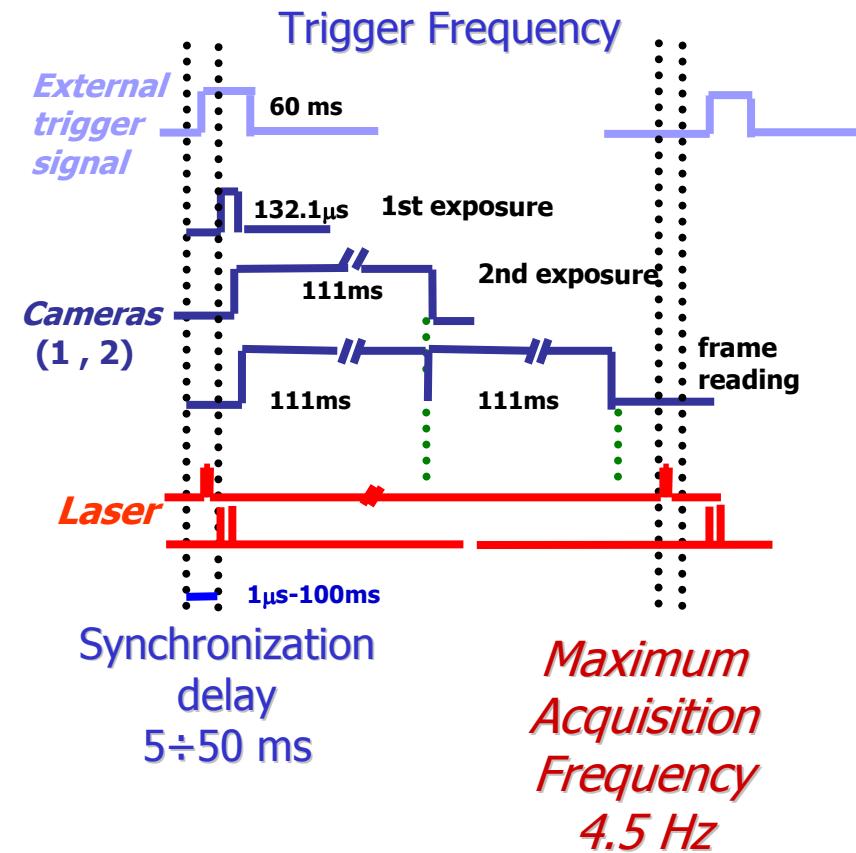
## 3D PIV Principle



## PIV System

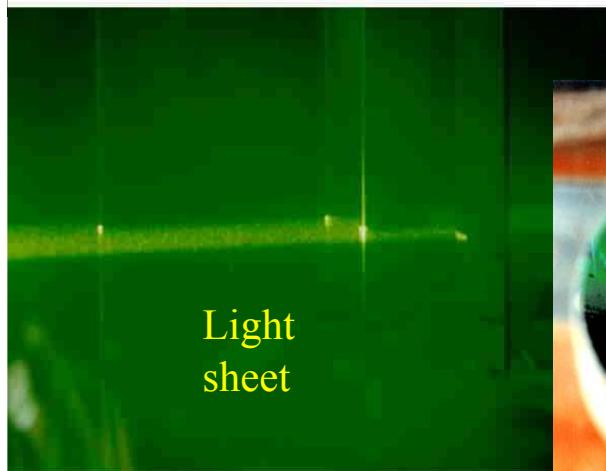
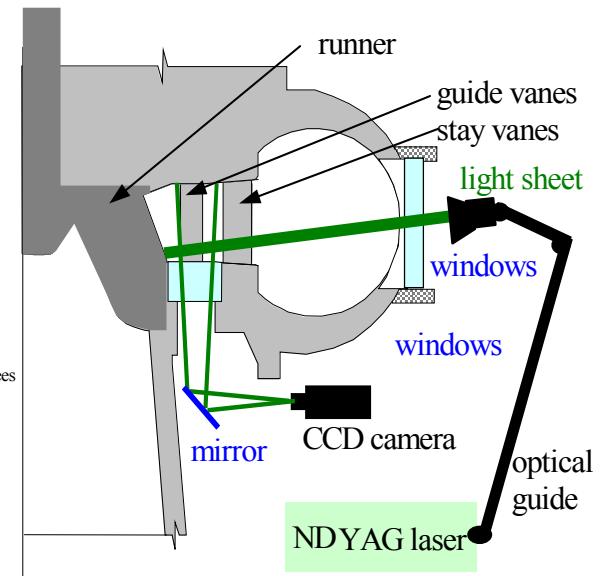
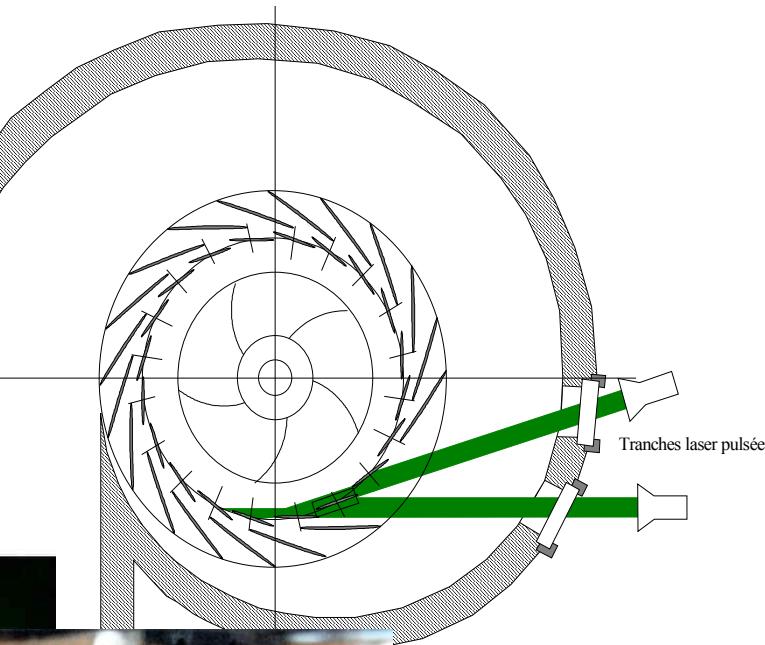
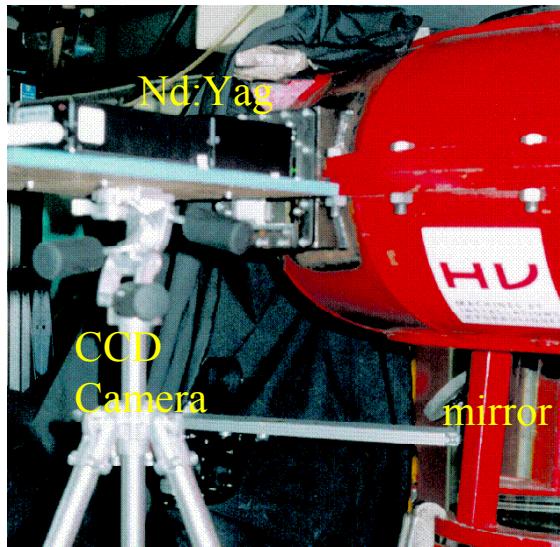


*System Architecture*

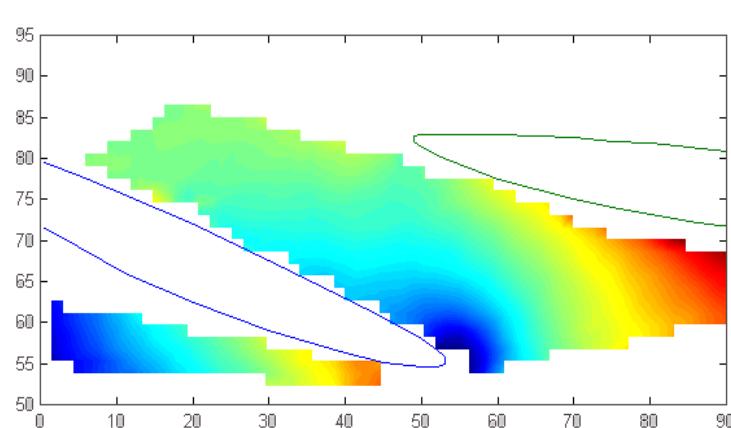


*Timing Diagram*

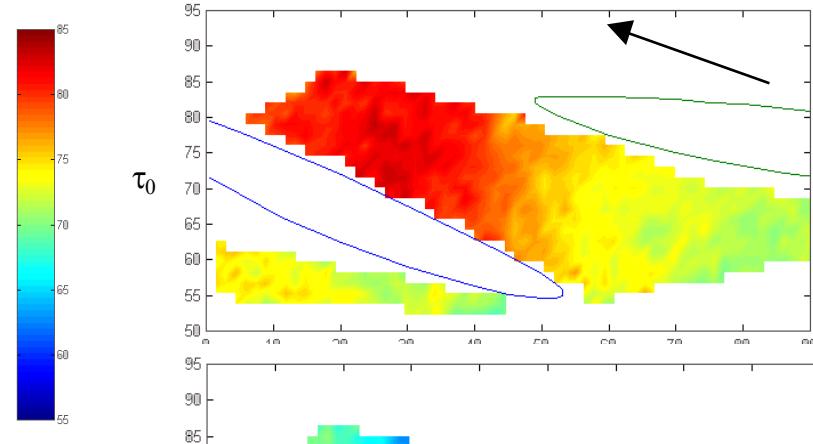
## Rotor-Stator Interaction



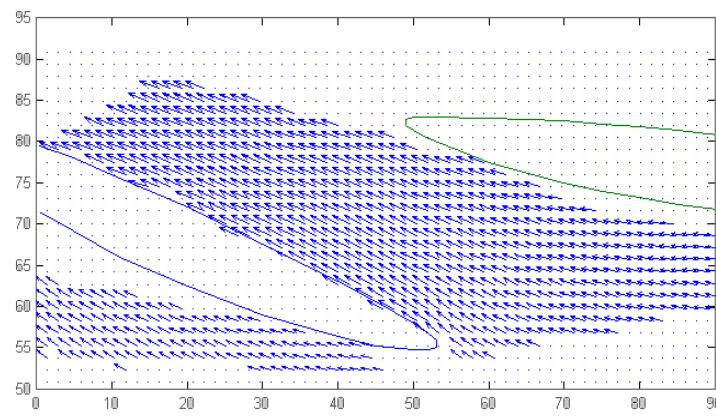
## Turbine Rotor-Stator Interaction



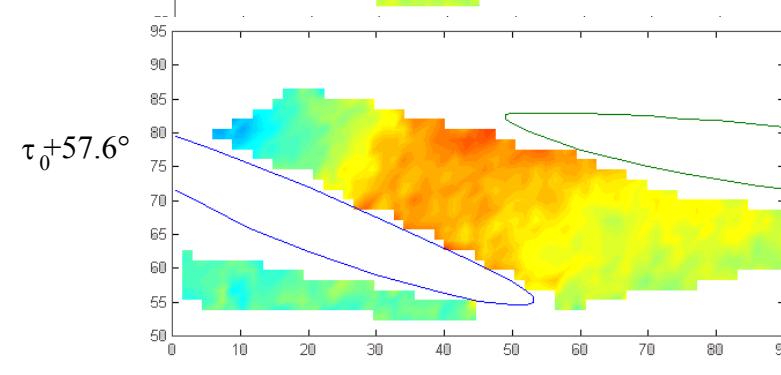
Angle of the Velocity Vector



$\tau_0 + 28.8^\circ$

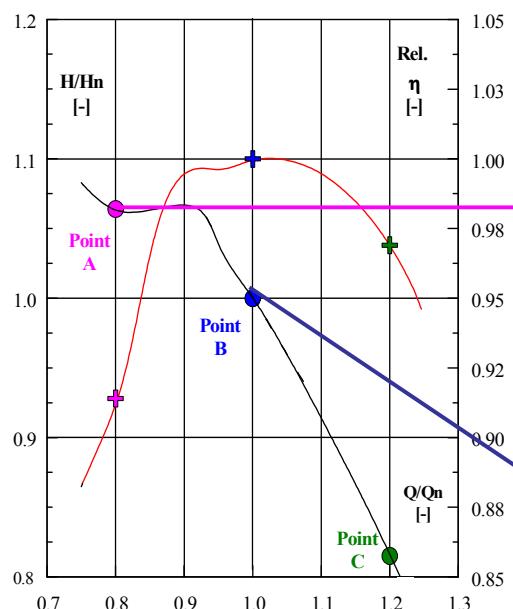


Mean Velocity Field

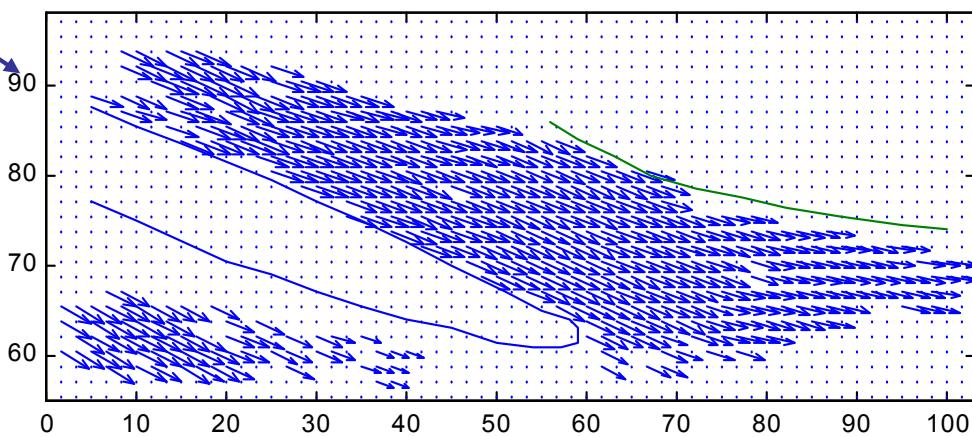
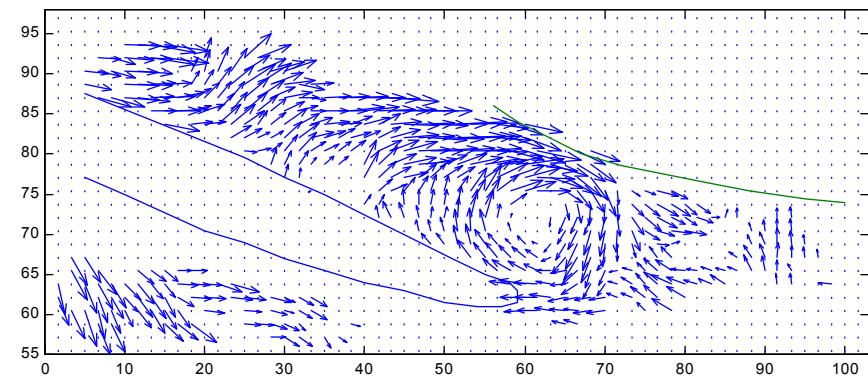


$\tilde{c} - \bar{c}$





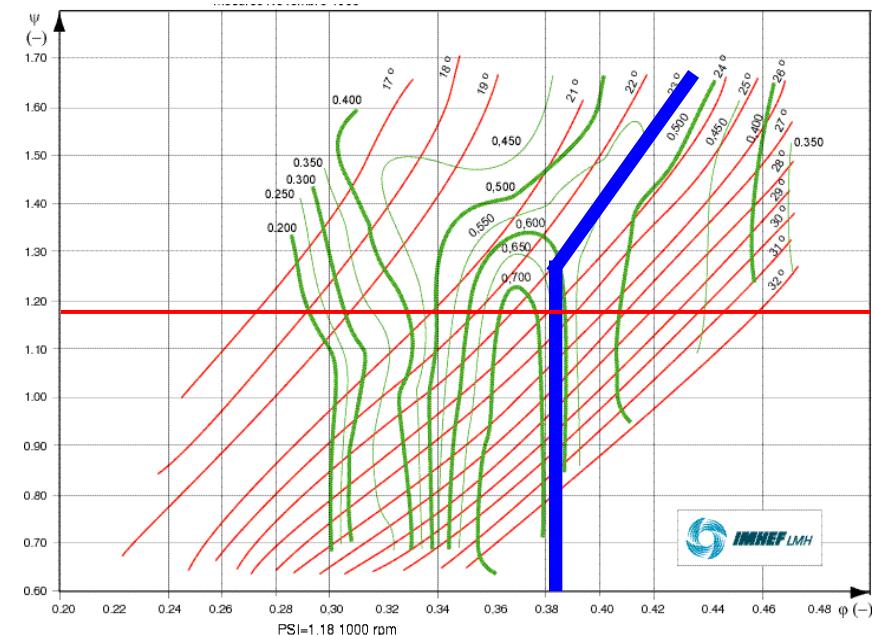
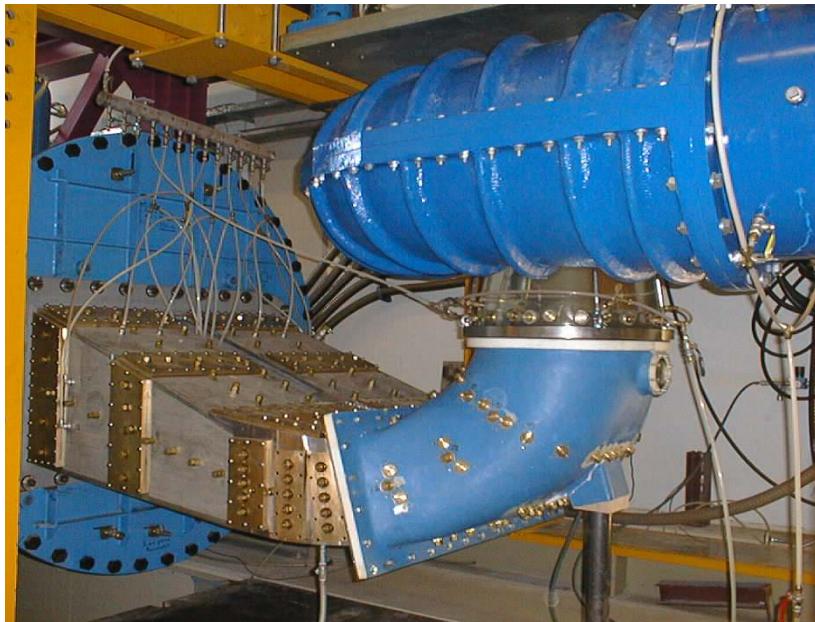
## Pump Rotor-Stator Interaction



Ciocan G.D., Avellan F., Kueny J.L.; "Optical Measurement Techniques for Experimental Analysis of Hydraulic Turbines Rotor - Stator Interaction"  
ASME Fluids Engineering Division Summer Meeting, 2000

# *Metode moderne pentru cercetarea in Masini Hidraulice*

## FLINDT Project



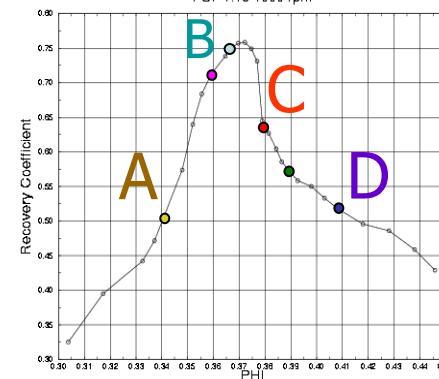
**ALSTOM**  
Power Hydro

**EDF**  
Electricité  
de France

**Power Systems**  
**GE Hydro**

**VA TECH**  
VA TECH HYDRO

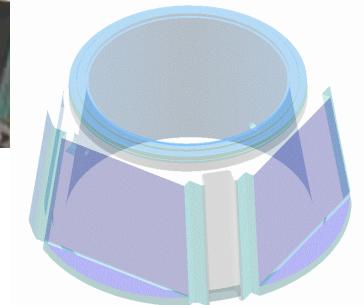
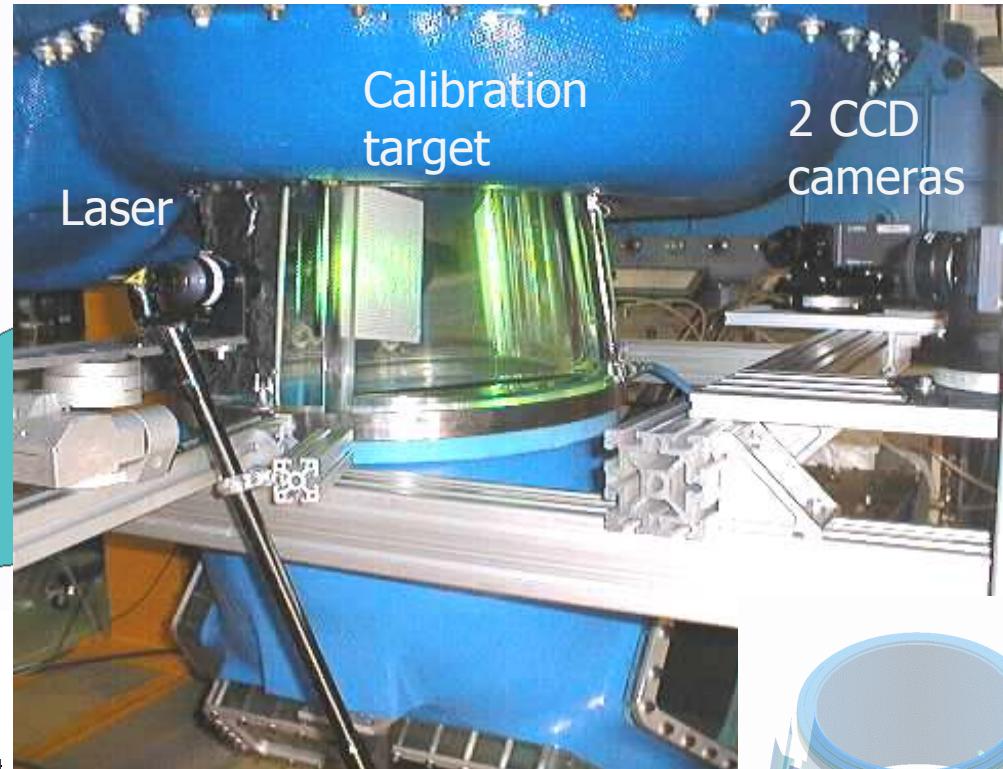
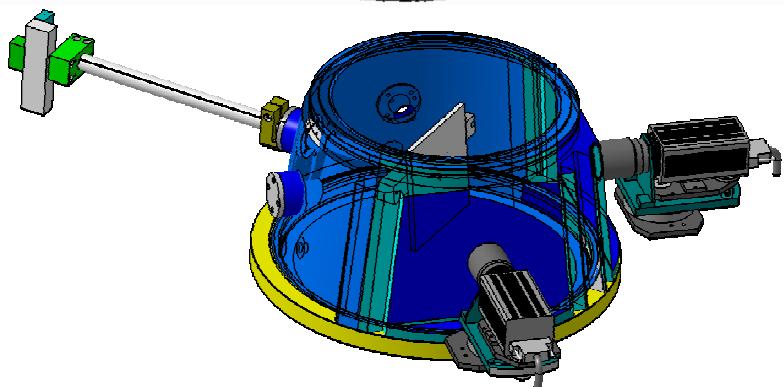
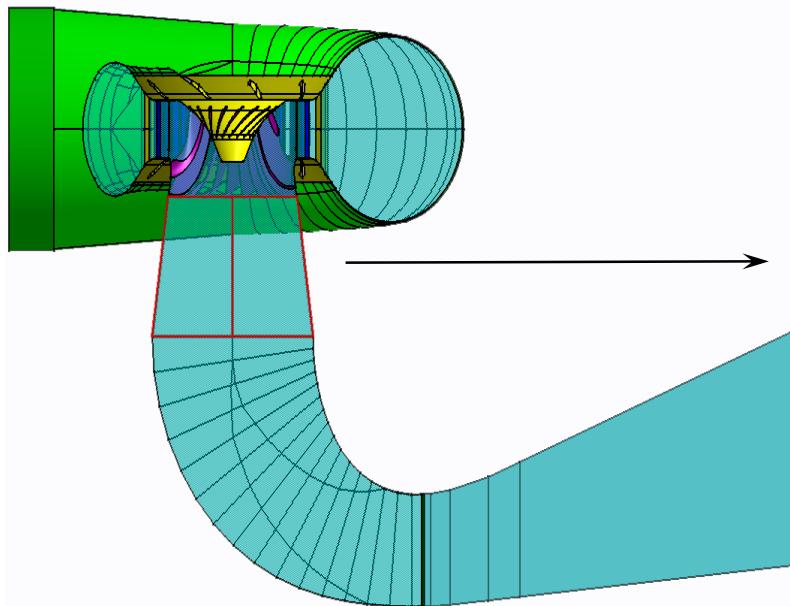
**VOITH SIEMENS**  
HYDRO POWER GENERATION



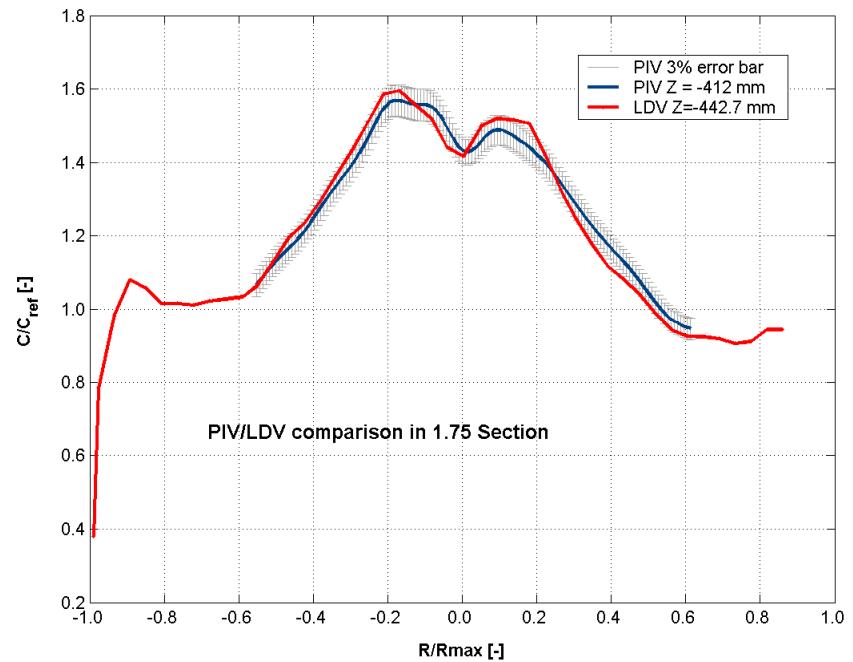
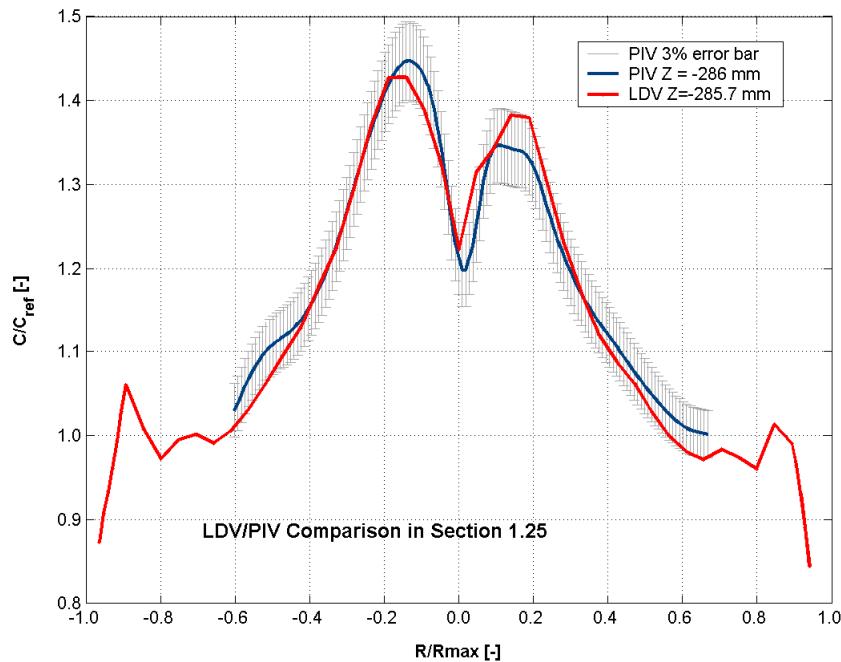
$$\chi = \frac{\frac{1}{\rho} \Delta p_{asp}}{\frac{1}{2} \left( \frac{Q}{A_{ref}} \right)^2}$$

Ciocan G.D., Avellan F., "Flow Investigations in a Francis Draft Tube: Advanced Experimental Methods"  
3rd Conference of Romanian Hydropower Engineers, Romania, 2004

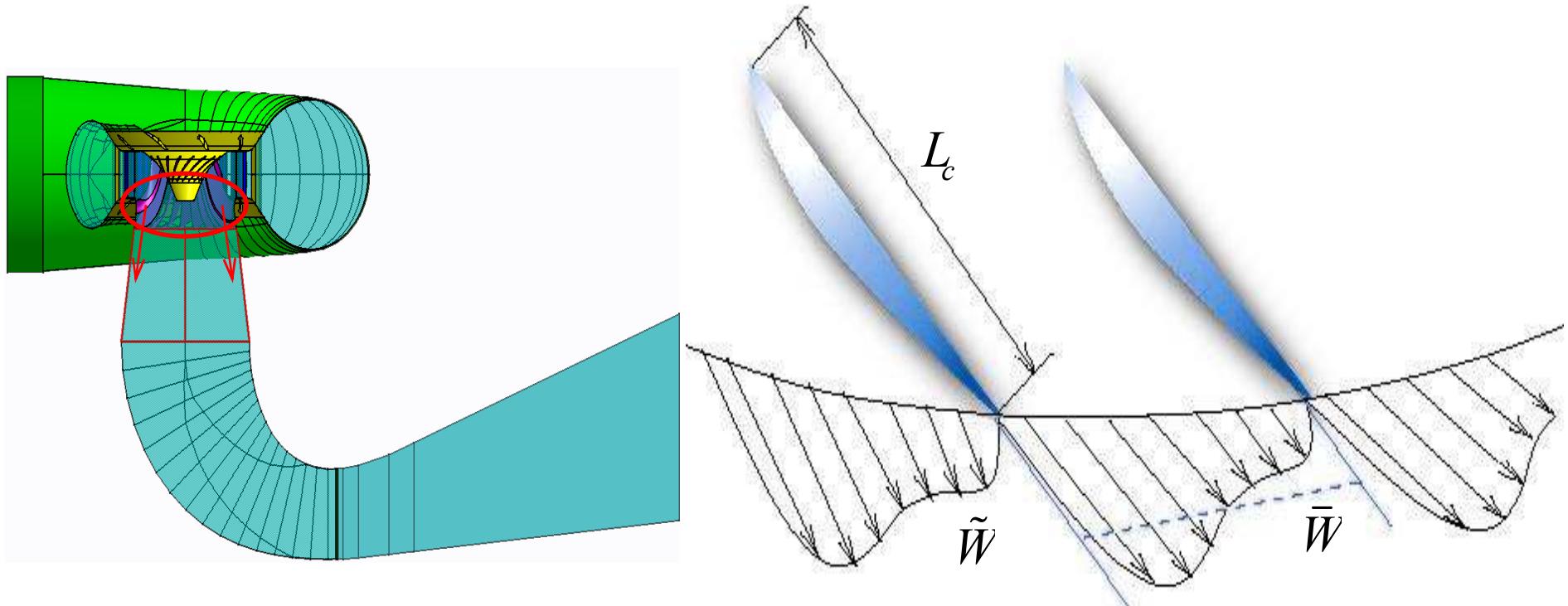
## 3D PIV in the Cone



## 3D Particle Image Velocimetry

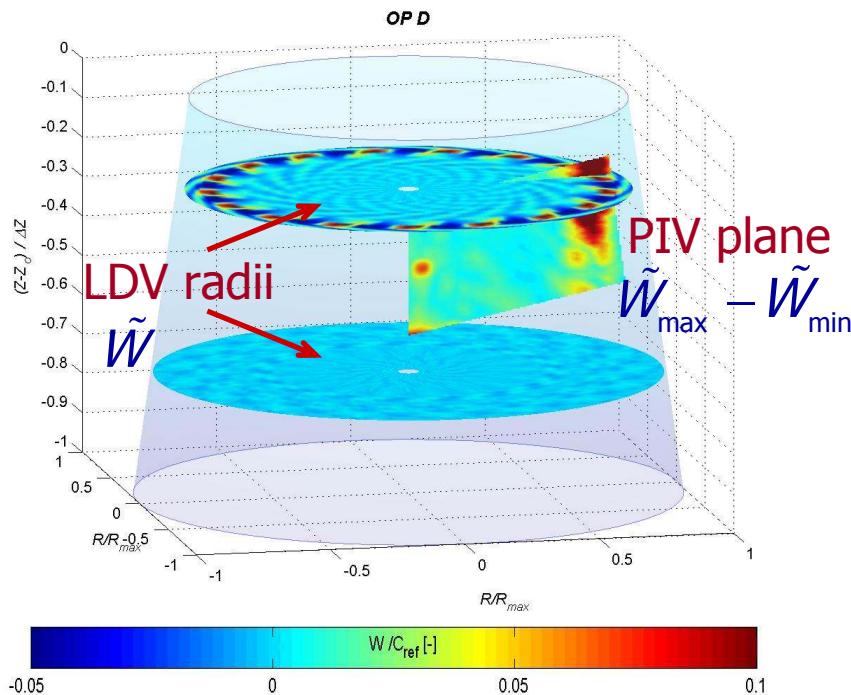


## Sheared flow mixing



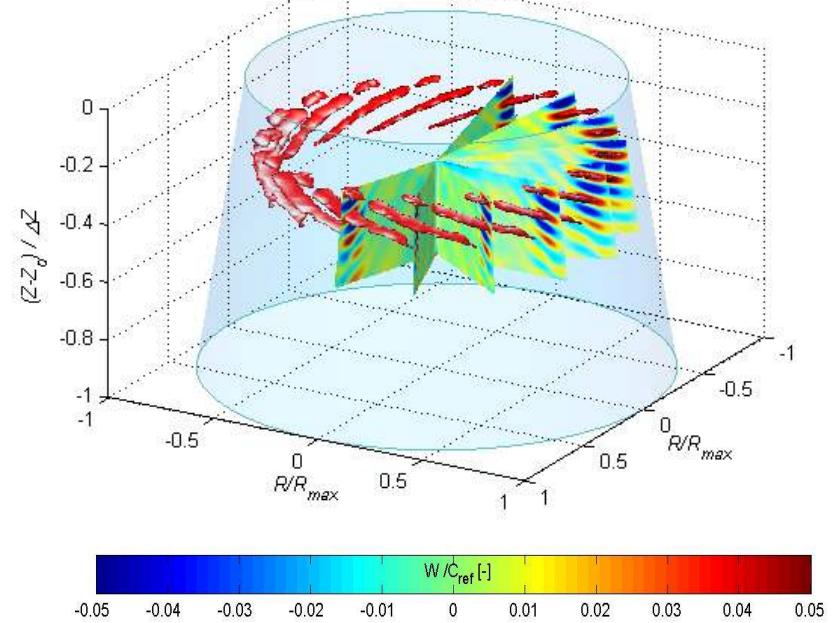
- Blade-to-blade flow field non-uniformity
- Pressure/suction side boundary layers shearing
- Led by the relative flow

## Sheared flow mixing



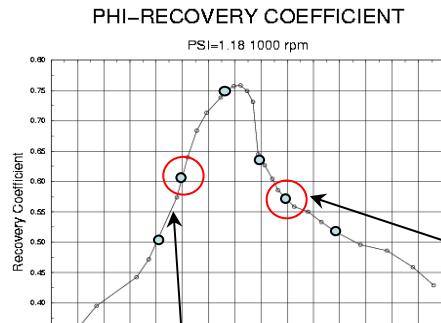
LDV – phase average velocity fields  
 PIV – fluctuation amplitude field

$$(\tilde{W}_{\max} - \tilde{W}_{\min}) / C_{\text{ref}} = k(L/L_c)^{-1}$$



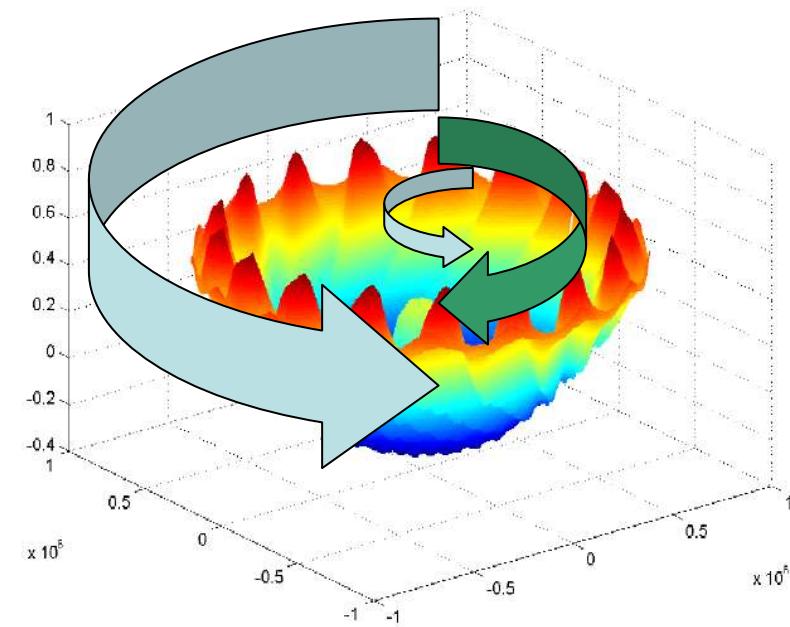
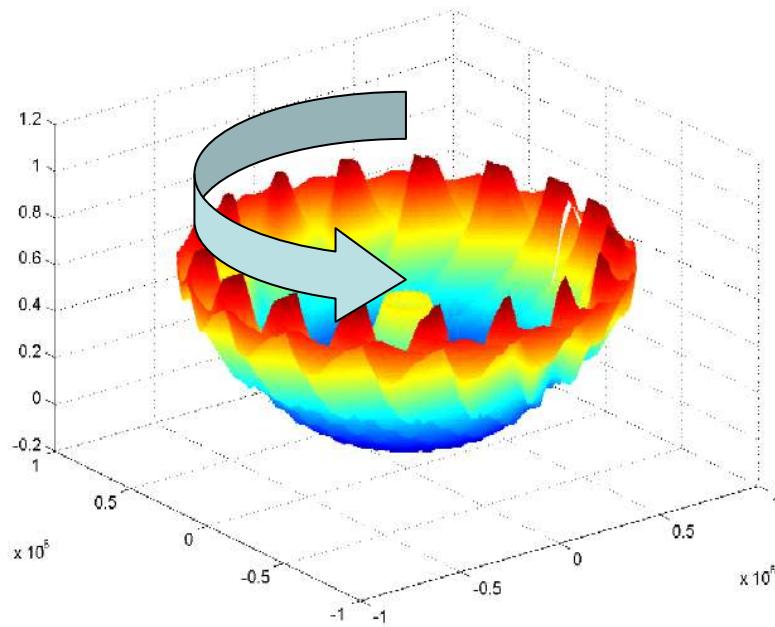
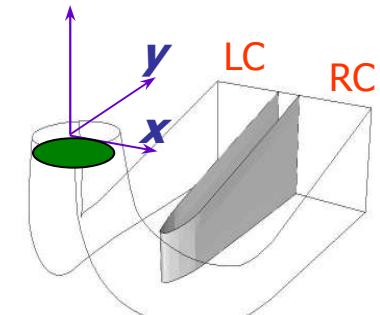
Blade-to-blade channel flow velocity  
 propagation & dispersal in the cone

## LDV Runner Outlet – Phase Average



Unsteady  
values

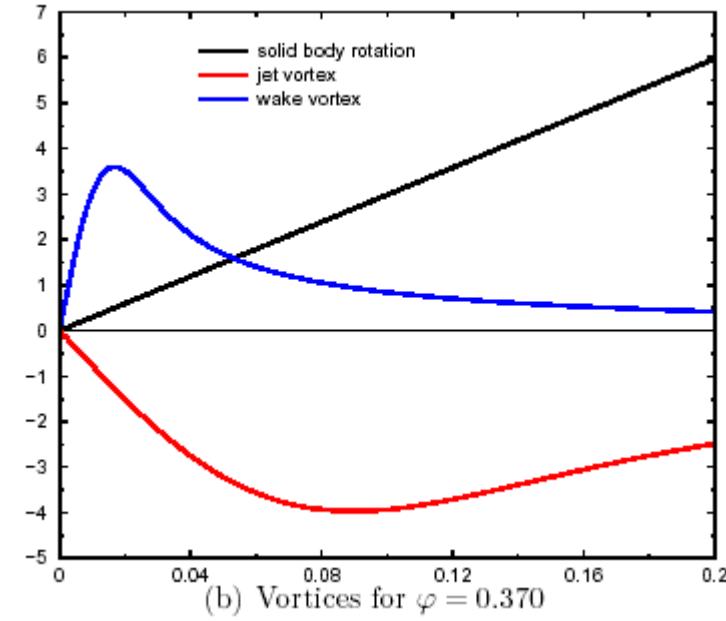
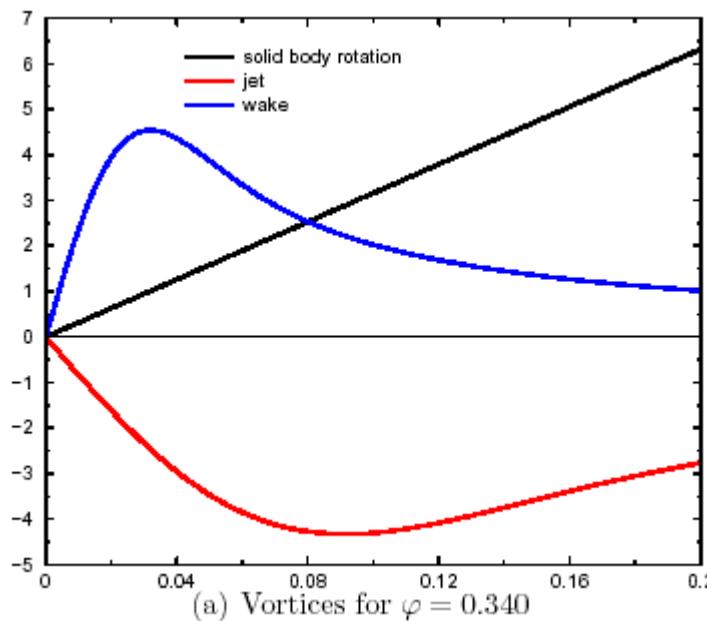
$$\tilde{c}_w^T + \bar{c}_w$$



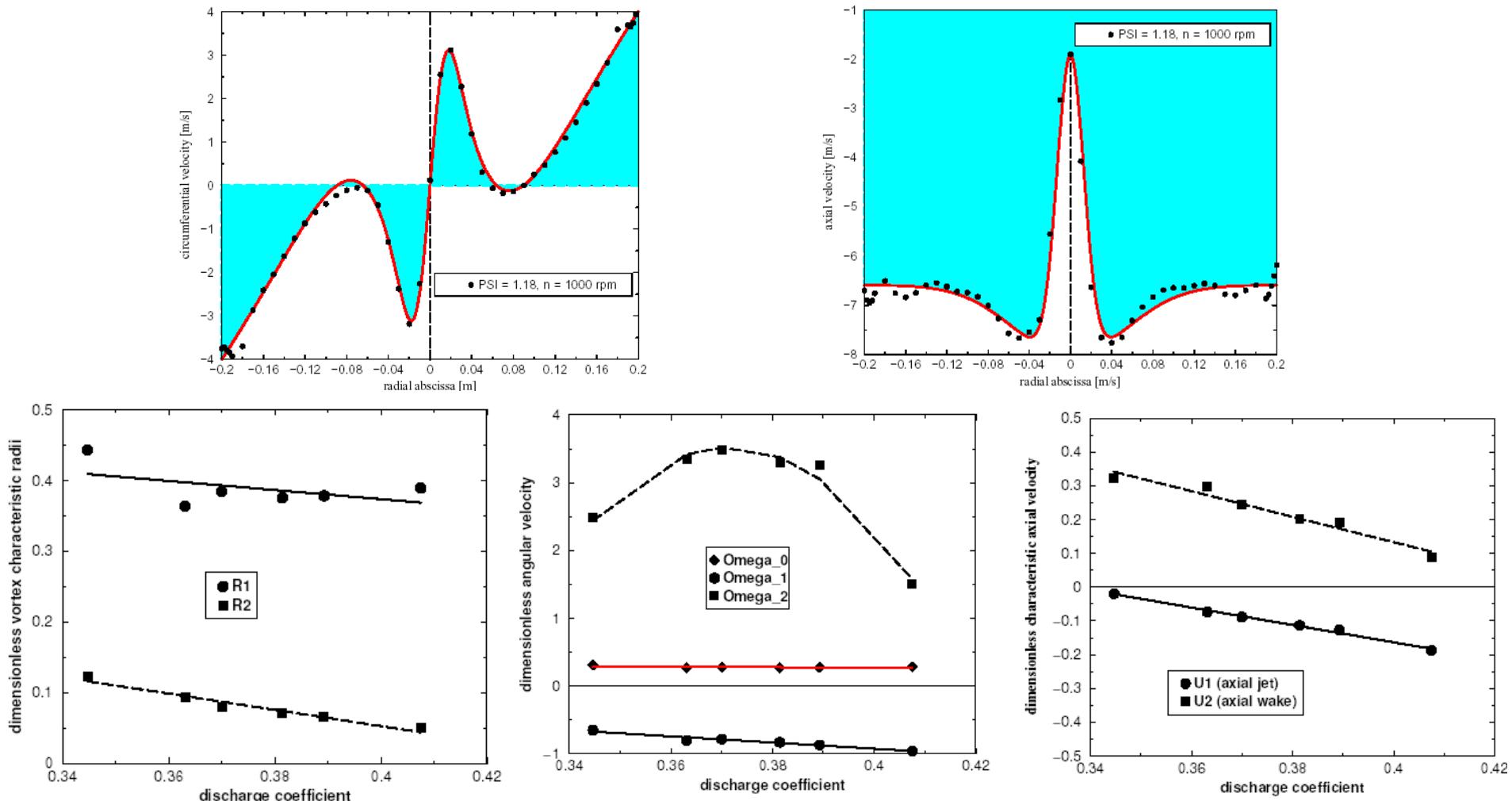
## Analytical Description of the Velocity Profile

$$v_u = \Omega_0 r + \Omega_1 \frac{R_1^2}{r} \left[ 1 - \exp \left( -\frac{r^2}{R_1^2} \right) \right] + \Omega_2 \frac{R_2^2}{r} \left[ 1 - \exp \left( -\frac{r^2}{R_2^2} \right) \right] \quad (1)$$

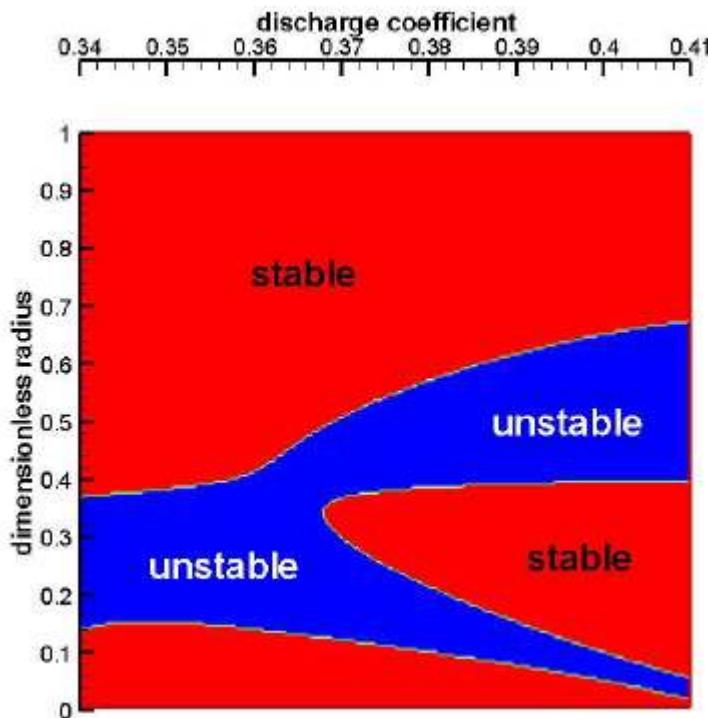
$$v_a = U_0 + U_1 \exp \left( -\frac{r^2}{R_1^2} \right) + U_2 \exp \left( -\frac{r^2}{R_2^2} \right) \quad (2)$$



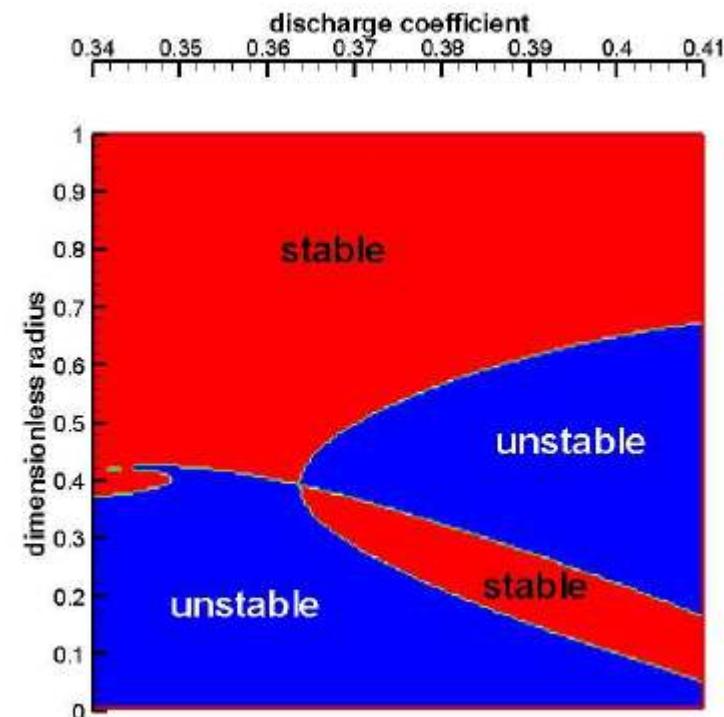
## Analytical Description of the Velocity Profile



## Étude de stabilité du profile de vitesse



Critère de stabilité Howard-Gupta  
Perturbation axisymétrique



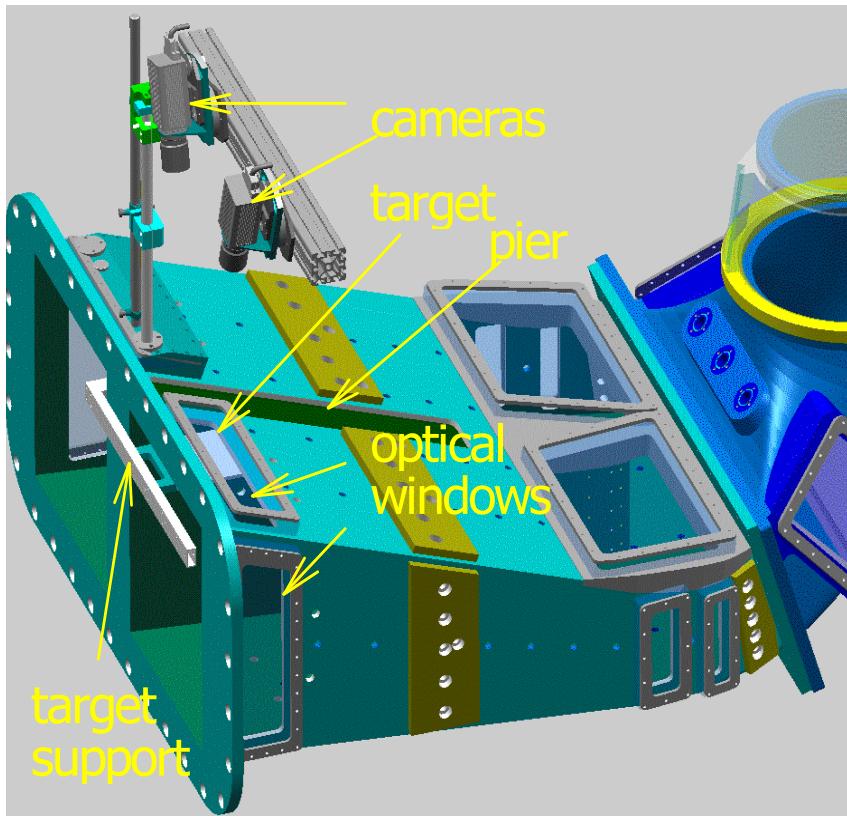
Critère de stabilité Leibovich-Stewartson  
Perturbation non axisymétrique

## Accuracy

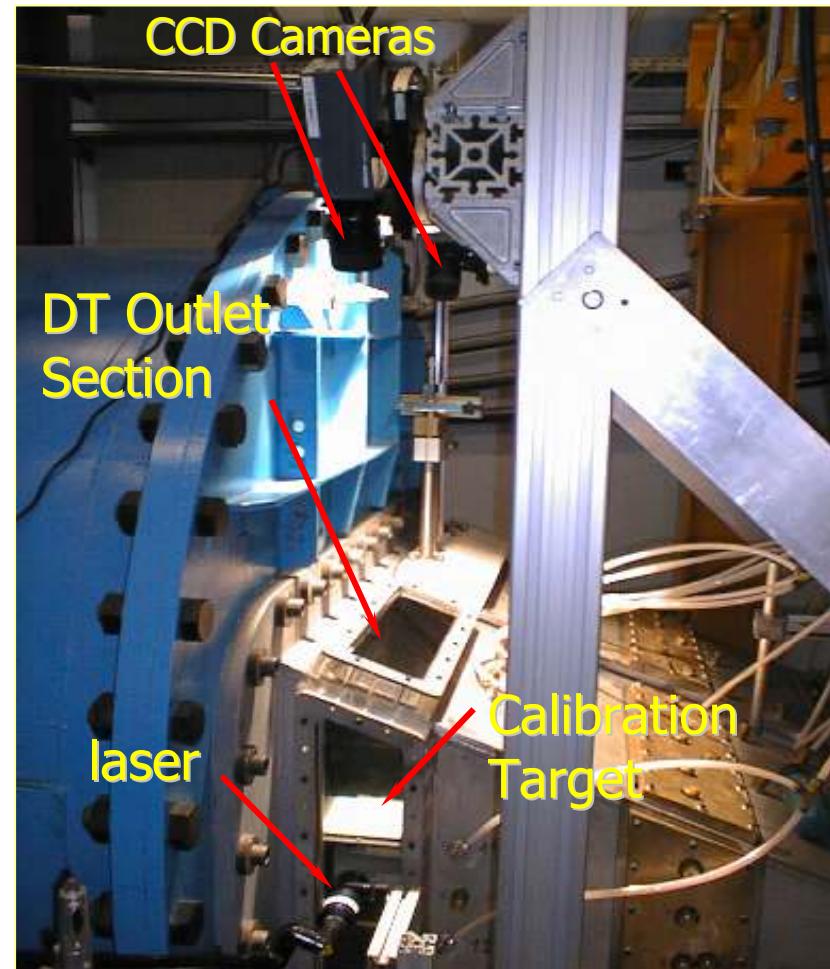
Measured discharge	Energ y Coeff.	Speed [rpm]	Computed discharge	Error [%]
0.340	1.18	1000	0.344	+1.1
0.360	1.18	1000	0.363	+0.8
0.368	1.18	1000	0.372	+1.0
0.380	1.18	1000	0.381	+0.2
0.390	1.18	1000	0.389	-0.2
0.410	1.18	1000	0.409	-0.3
0.368	1.00	1000	0.368	+0.1
0.380	1.00	1000	0.380	+0.1
0.370	1.11	1000	0.369	-0.1
0.368	1.30	1000	0.371	+0.8
0.380	1.30	1000	0.386	+1.5
0.410	1.30	1000	0.407	-0.6
0.370	1.11	500	0.370	+0.1
0.340	1.18	500	0.345	+1.6
0.368	1.18	500	0.369	+0.2
0.380	1.18	500	0.379	-0.3
0.410	1.18	500	0.406	-0.9

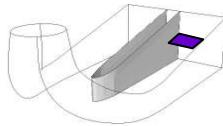
- Discharge computed from fitted axial velocity profile agrees within less than 1% with the measured discharge
- The energy coefficient does not influence the velocity profiles at constant discharge
- Reynolds number effect is negligible (same velocity profiles at different runner speed)

## 3D-PIV Draft Tube Outlet



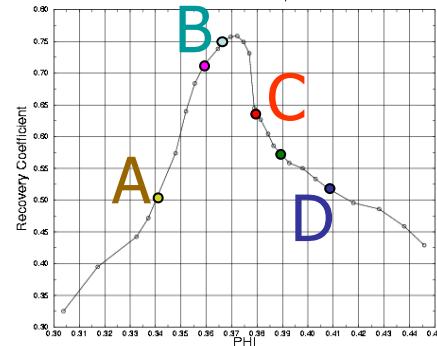
*Experimental Setup*



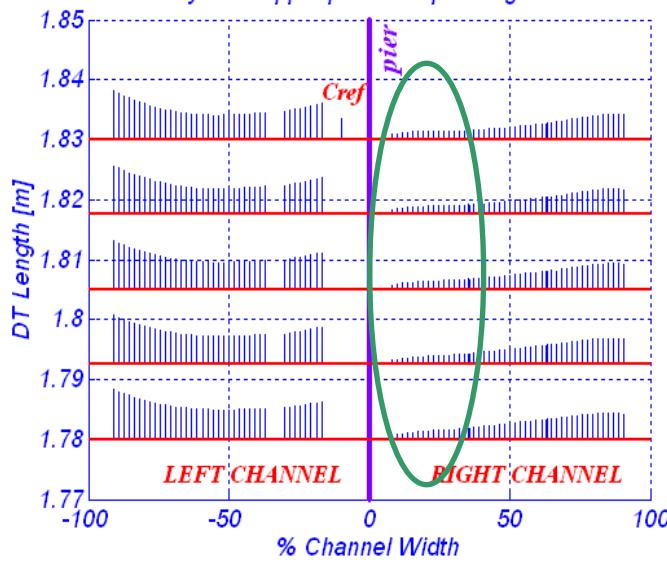


PHI-RECOVERY COEFFICIENT

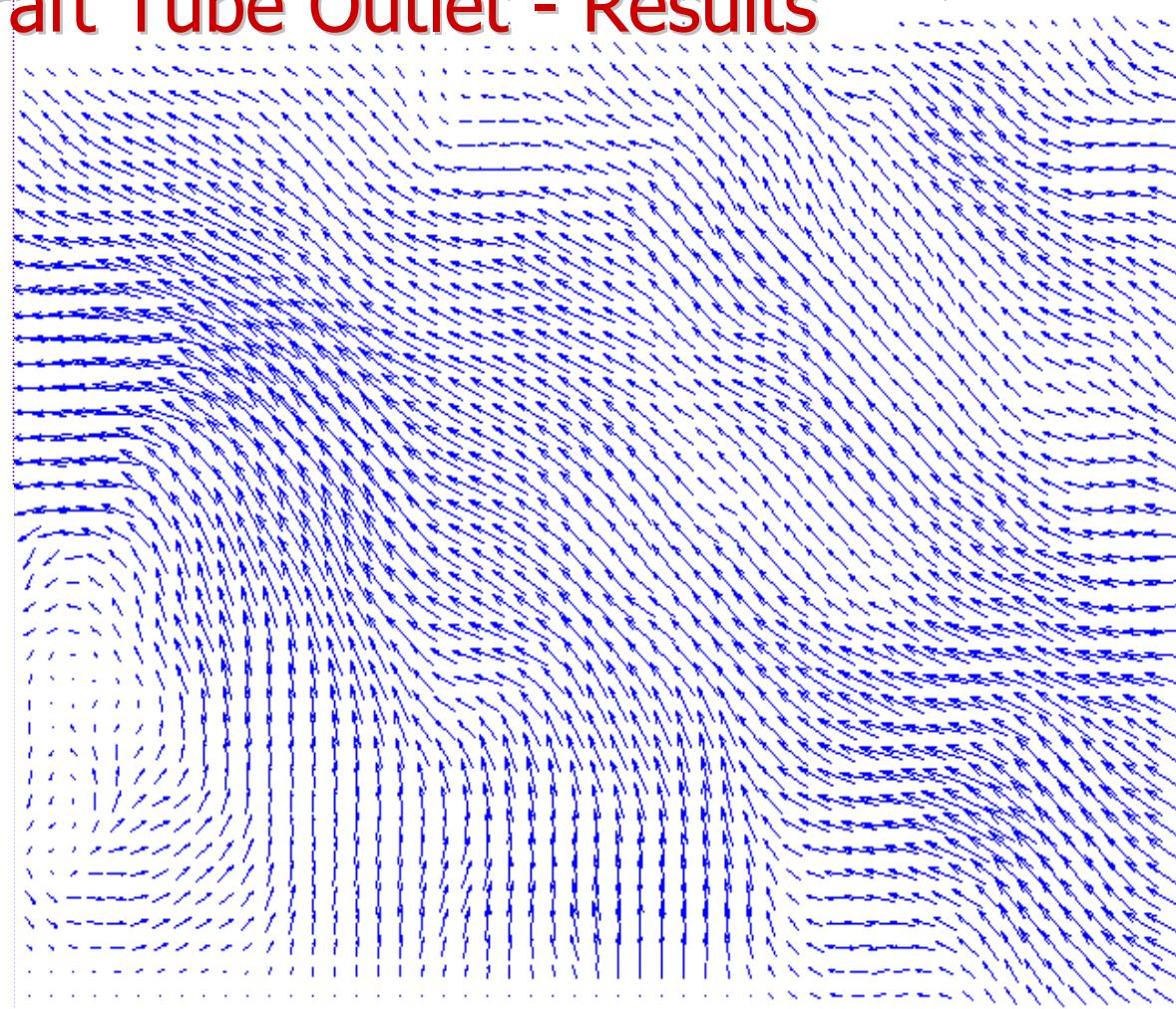
PSI=1.18 1000 rpm



Cx-Cy-Cz ~upper plane~ Operating Point D

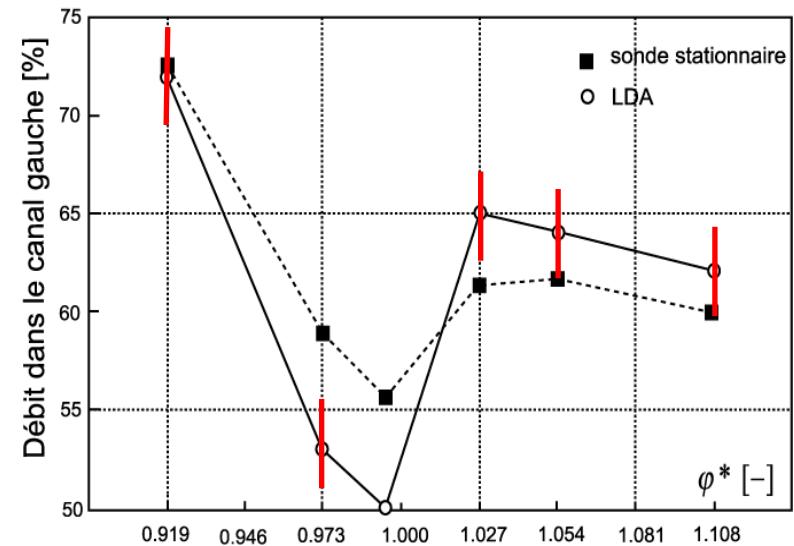
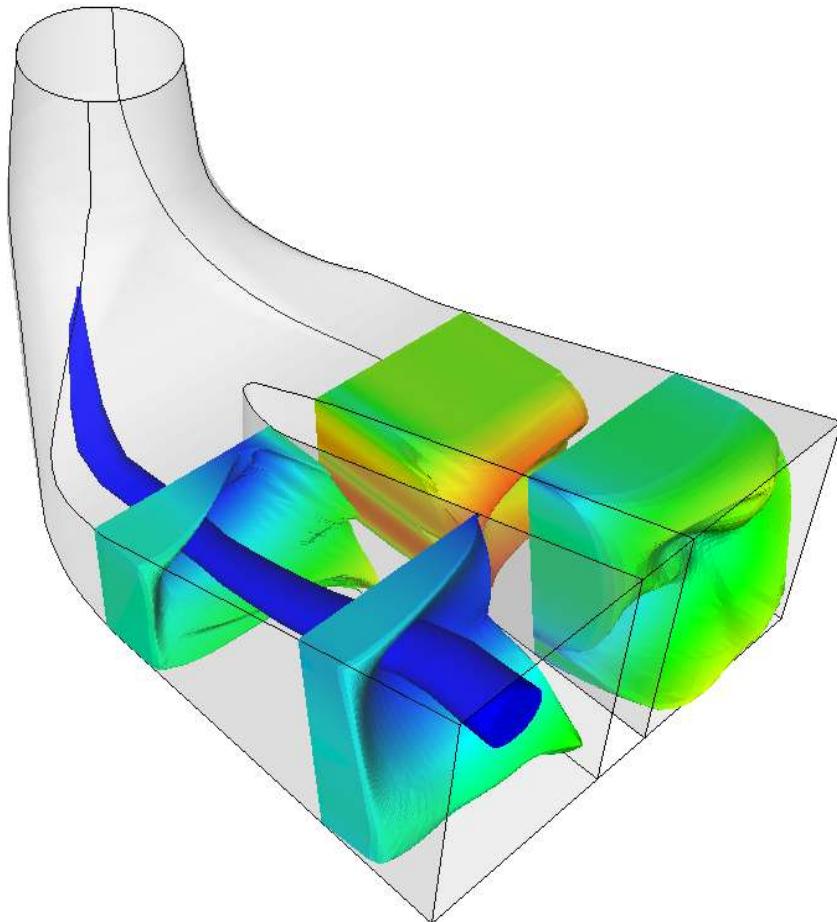


## 3D-PIV Draft Tube Outlet - Results



*Instantaneous 2D Vector Fields*

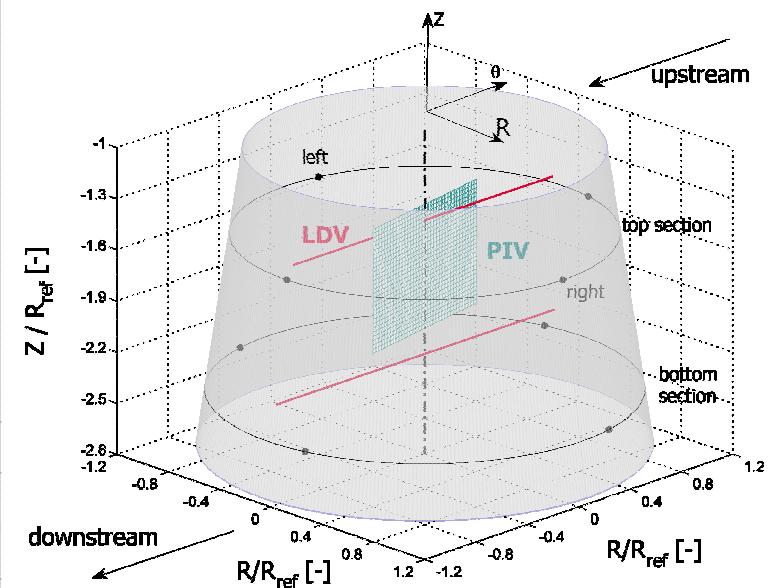
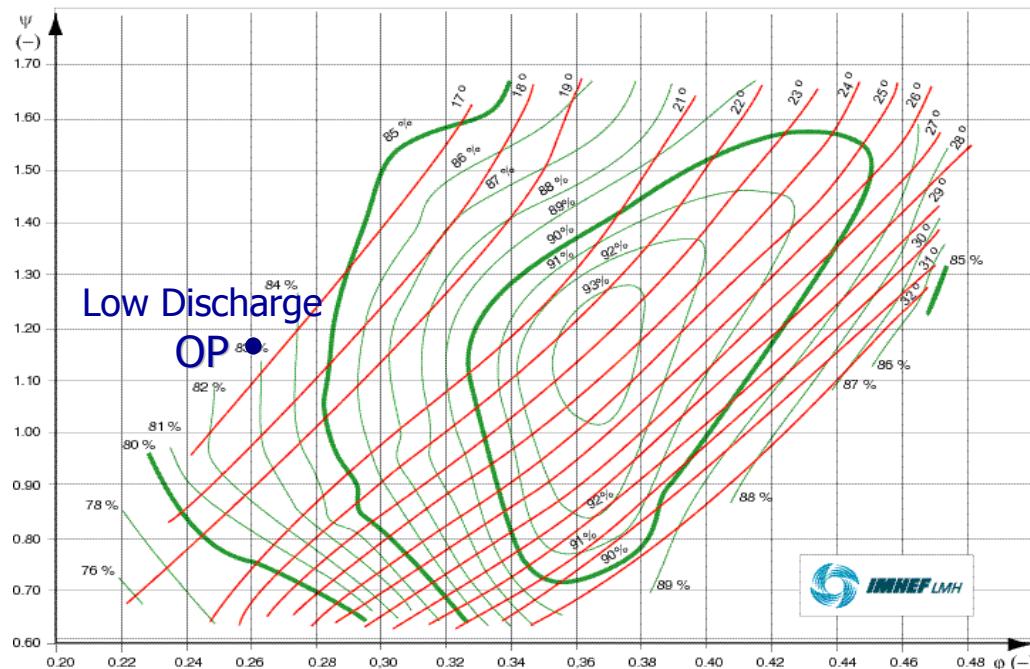
## Flow rate distribution



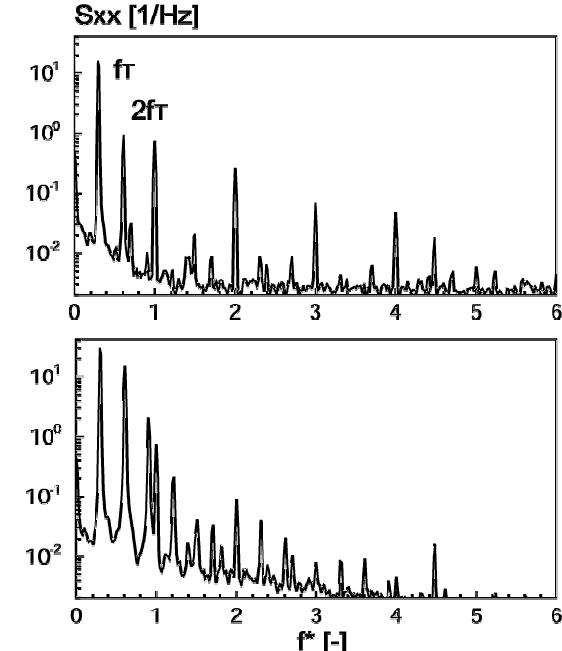
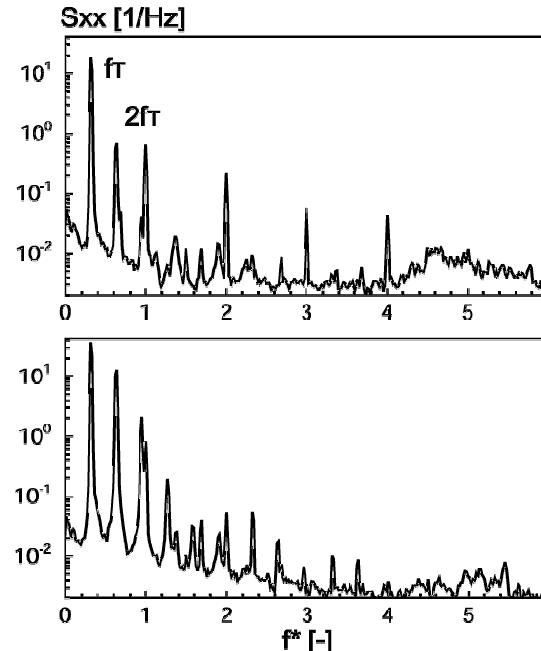
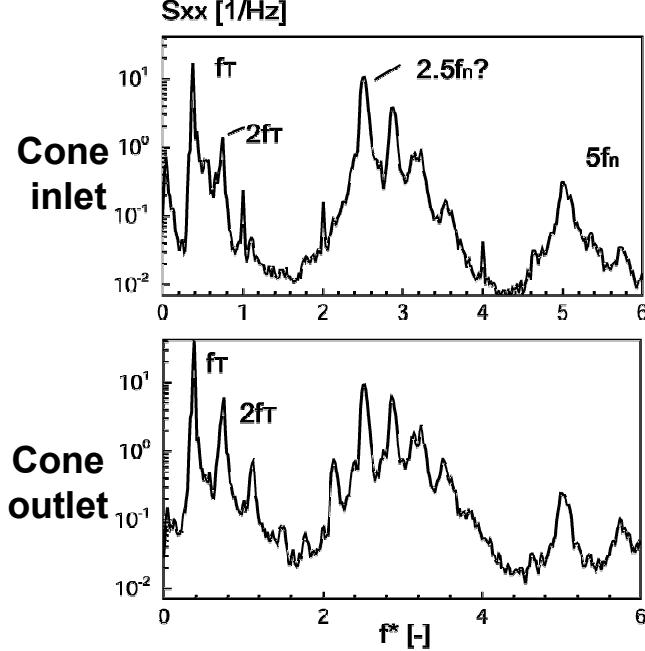
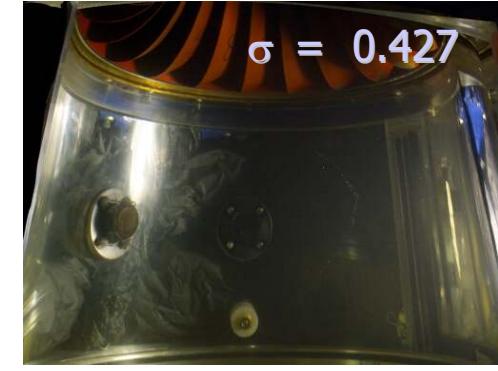
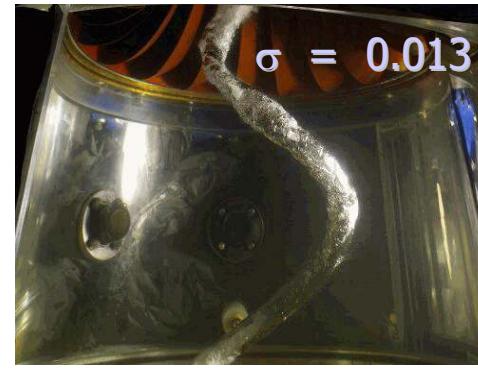
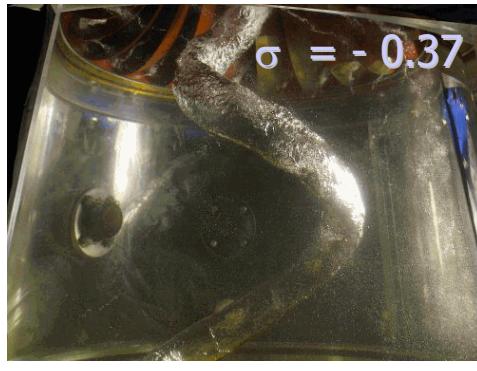
Accuracy of the flow rate estimation:  $\sim 4\%$

Iliescu M.S., Ciocan G.D., Avellan F., "3D PIV and LDV Measurements at the Outlet of a Francis Turbine Draft Tube"  
Joint US ASME-European Fluids Engineering Summer Conference, 2002

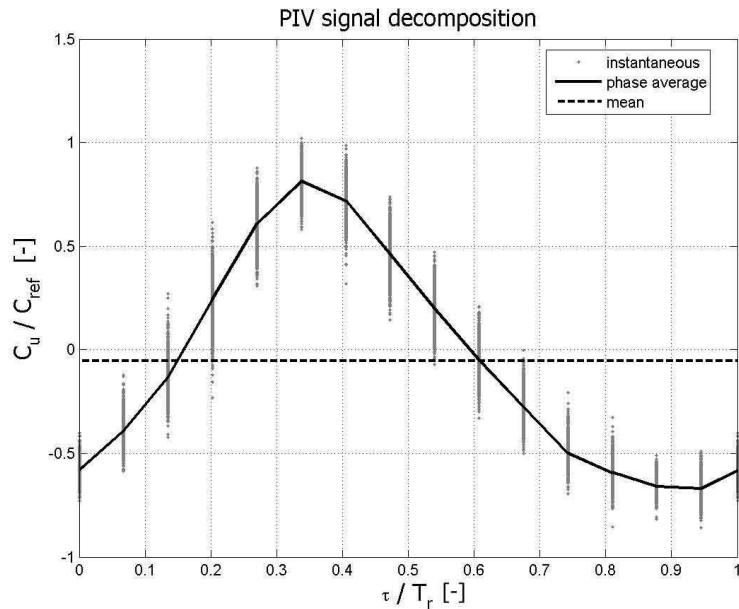
## Partial Flow Rate Operating Point



## Unsteady Pressure Field



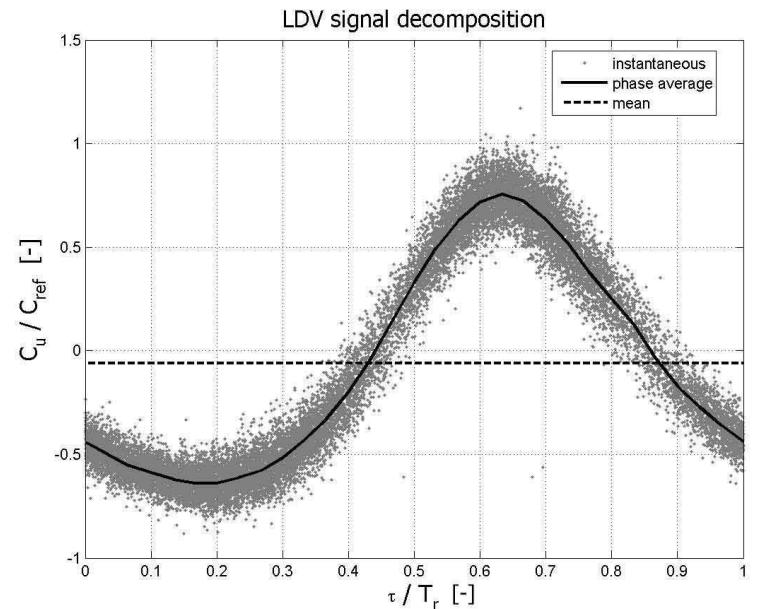
## Phase Average on the Rope Passage



Phase average - PIV

$$C_i(t) = \bar{C} + \tilde{C}(\tau) + C'(t)$$

$$\bar{C} = \langle C_i \rangle = \lim_{N \rightarrow \infty} \frac{1}{N} \sum_{i=1}^N C_i(t)$$



Phase average - LDV

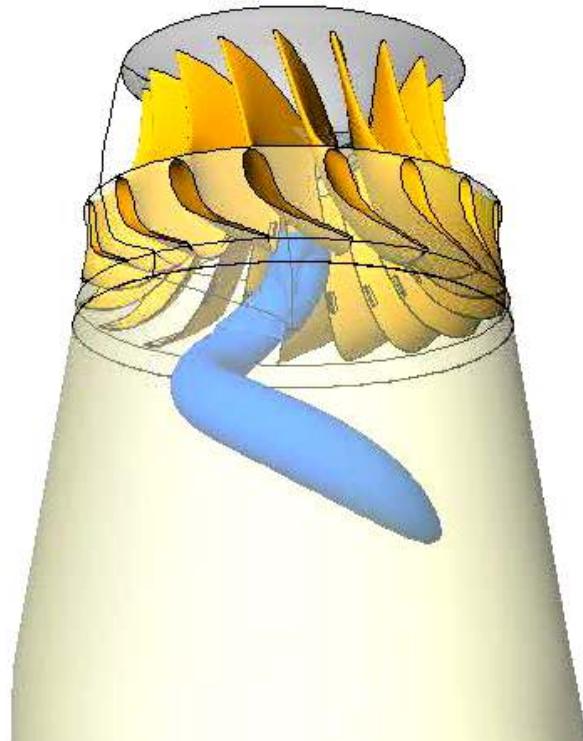
$$\tilde{C}(\tau) = \lim_{N \rightarrow \infty} \frac{1}{N} \sum_{i=1}^N (C_i(\tau) - \bar{C})$$

$$\bar{\tilde{C}} = 0$$

$$\langle C' \rangle = \lim_{N \rightarrow \infty} (C_i(t) - \bar{C} - \tilde{C}) = 0$$

## Unsteady Simulation of Rotating Rope

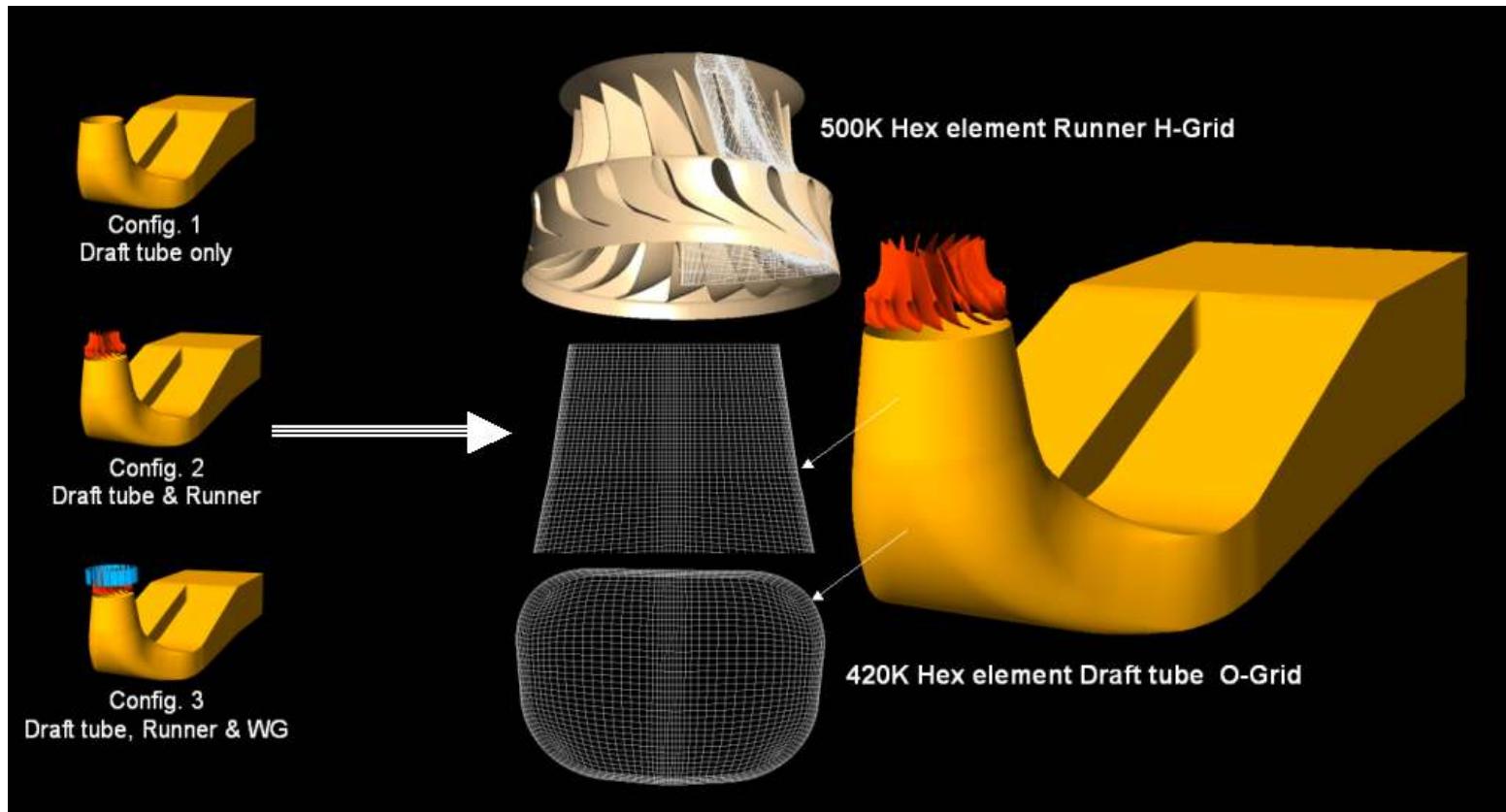
time = 0.00000



-Vortex rope without vapors

Ciocan G.D., Iliescu M.S., Vu T., Nennemann B., Avellan F., "Experimental Study and Numerical Simulation of the Flindt Draft Tube Rotating Vortex"  
Accepted for Journal of Fluids Engineering, 2005

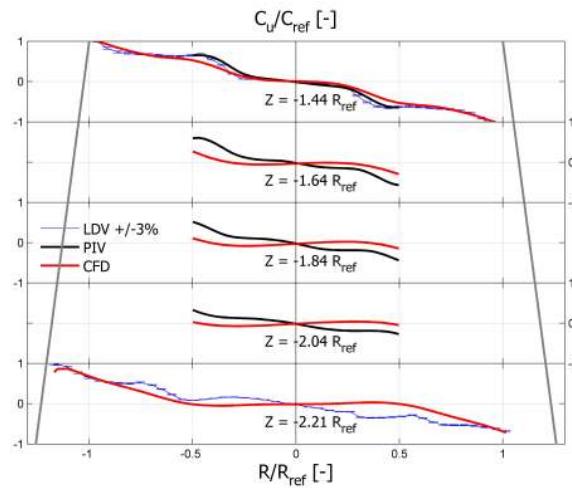
## Computation Flow Domain & Mesh



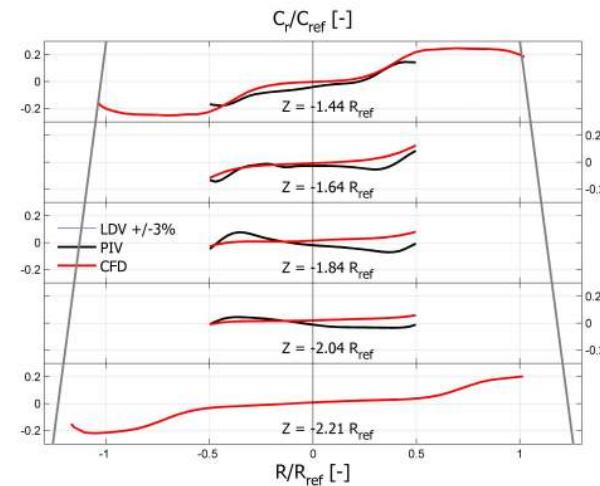
1.2 million node multi-block structured mesh  
Inlet condition obtained from previous stage  
calculation

## Average Velocity Profiles

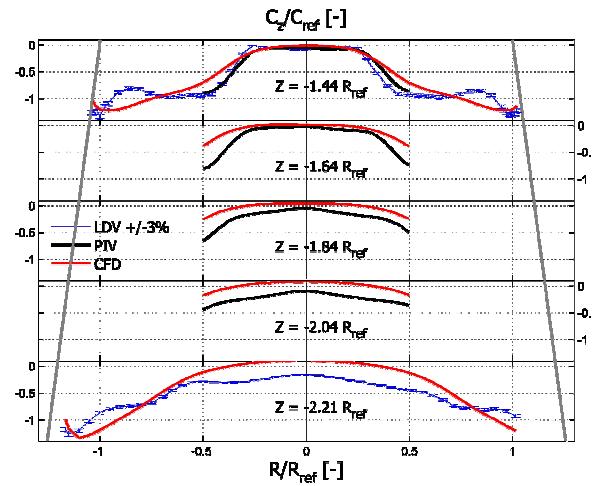
$C_u$



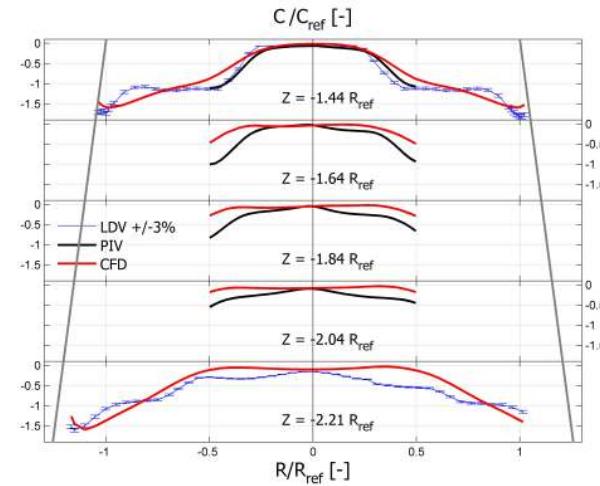
$C_r$



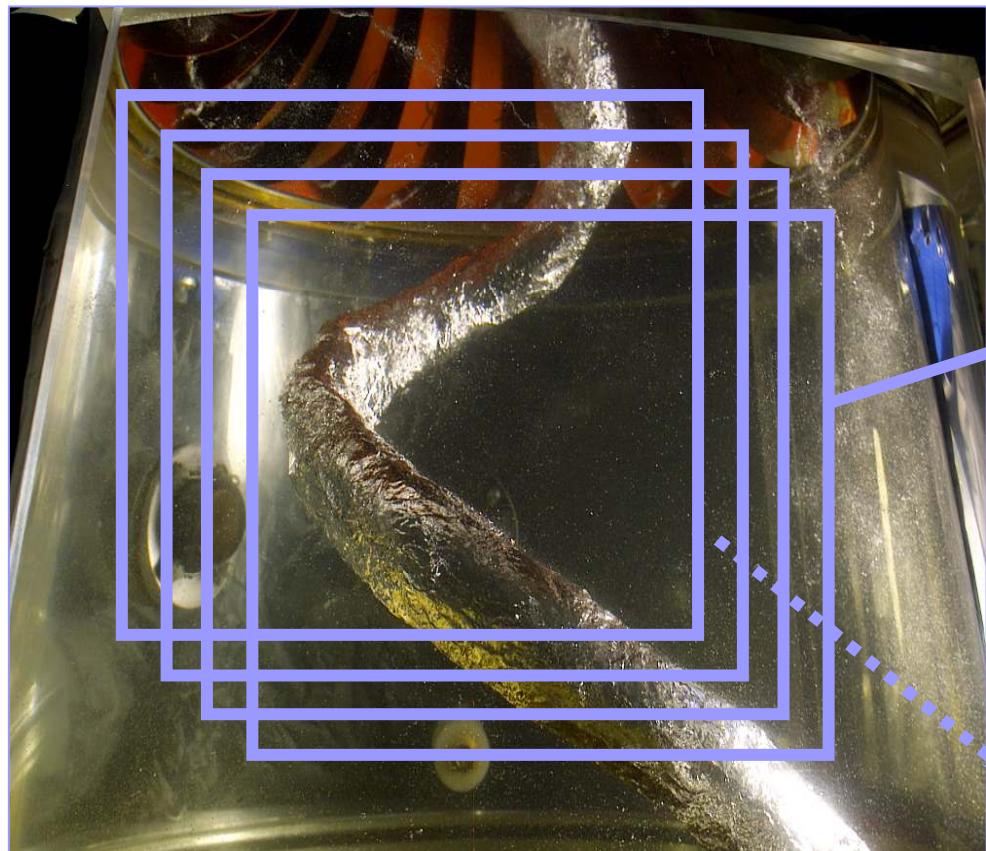
$C_z$



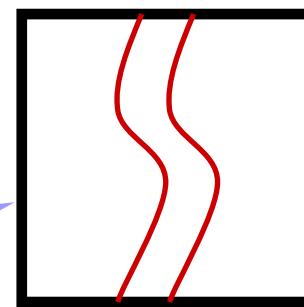
$C$



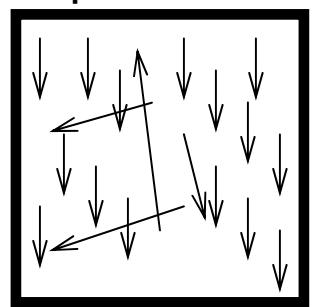
## PIV 2phases - Principle



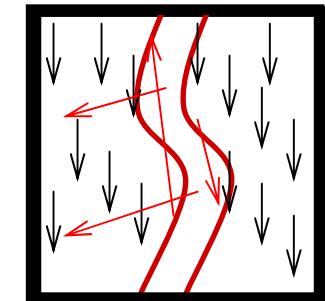
Green  
reflections



Fluorescent  
particles



7 positions X

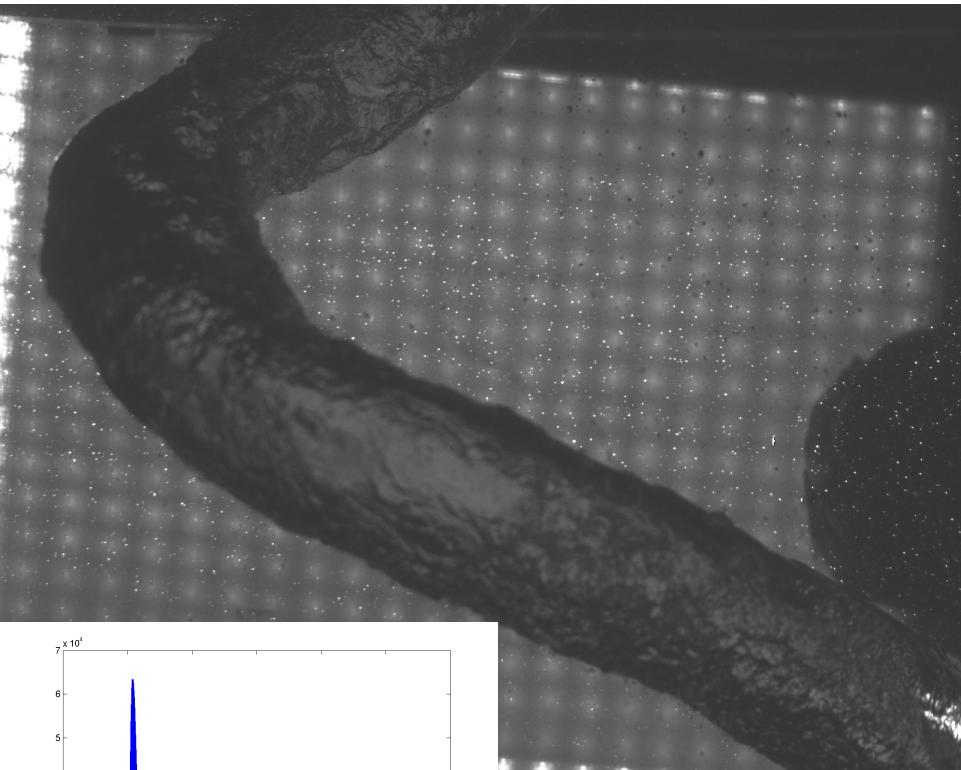


Synchronous with pressure signal for  
different azimuthal positions of the rope

*Rope shape  
+  
2D velocity field*

*Metode moderne pentru cercetarea in  
Masini Hidraulice*

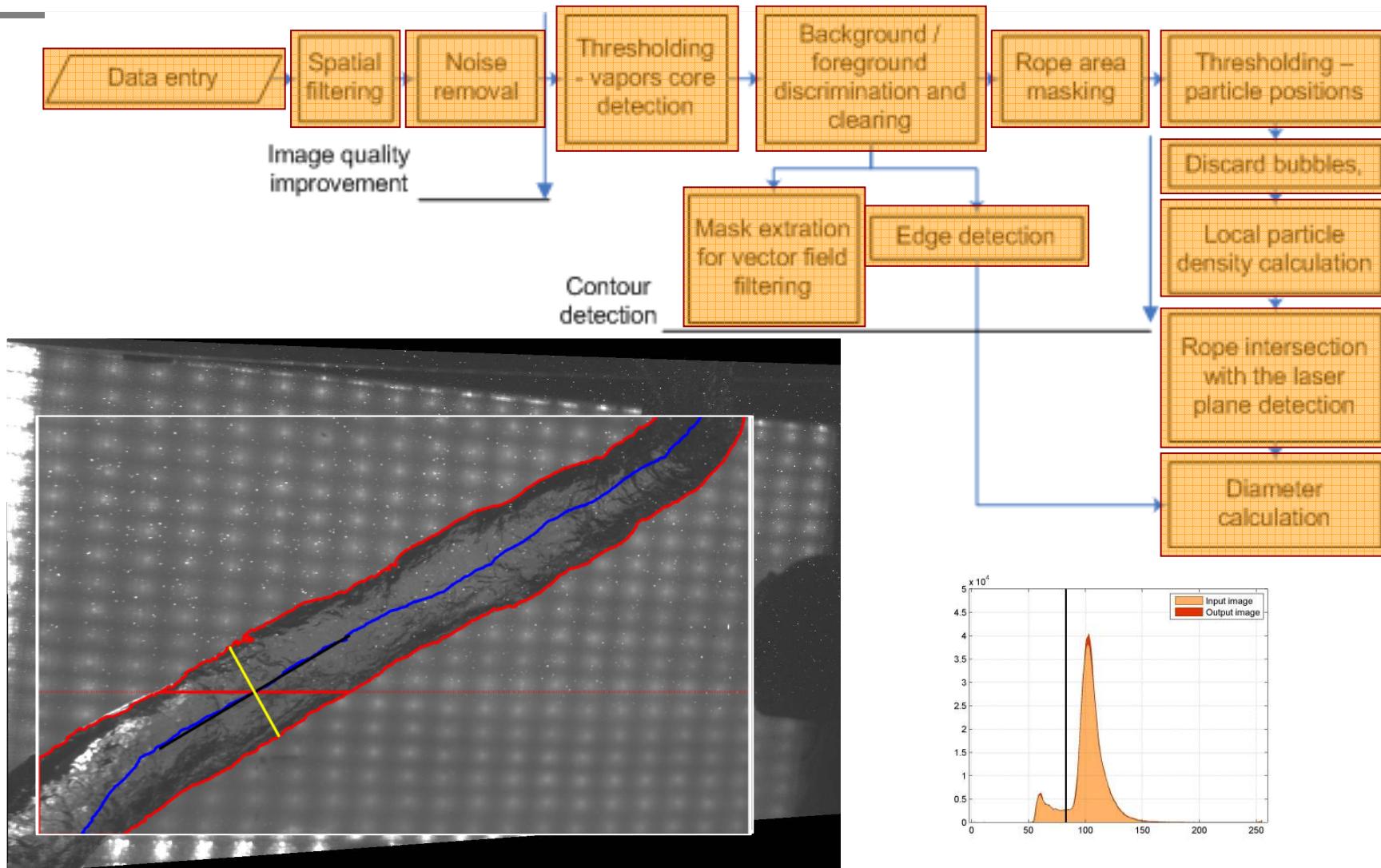
## 3D 2phase PIV



- Laser
- LED
- fluorescent particles
- filters

532 nm  
587 nm  
580 nm  
>570 nm

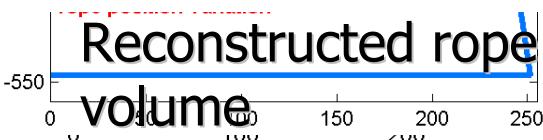
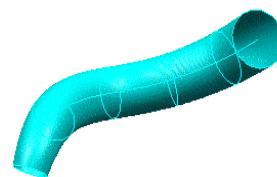
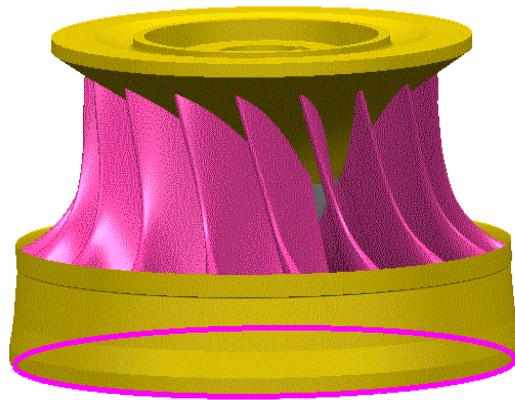
# Metode moderne pentru cercetarea in Masini Hidraulice



*Contour Validation Criteria*

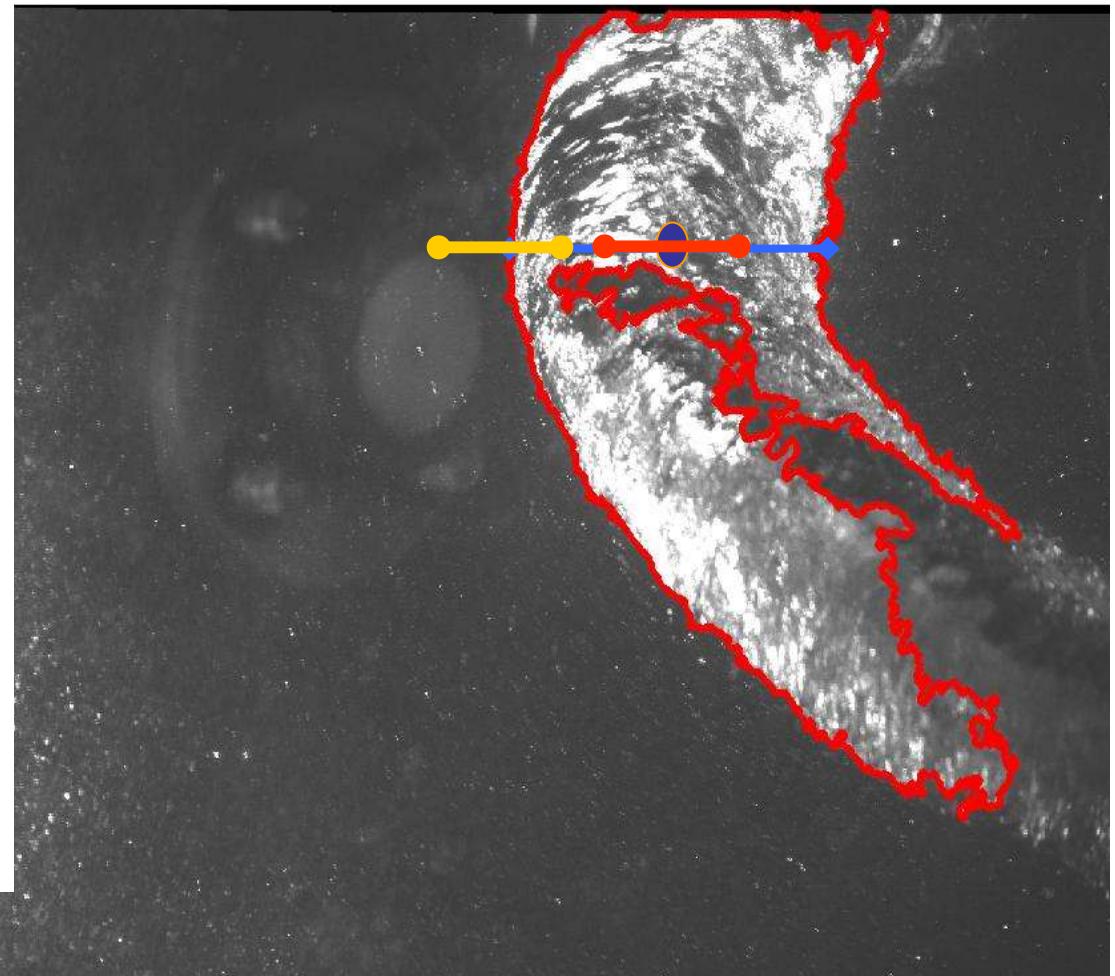
- ✓
- ✓

-20  
-25  
-30  
-35  
-40  
-45  
-50  
-550



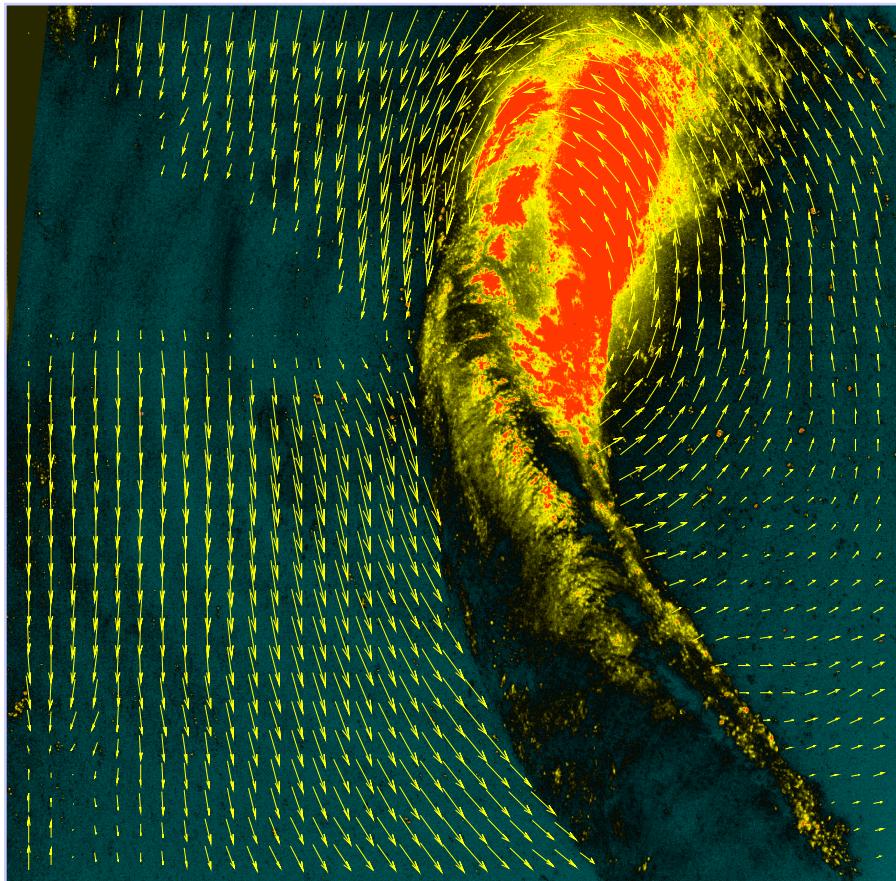
**Rope Dynamics**

More to our rope position fit situations



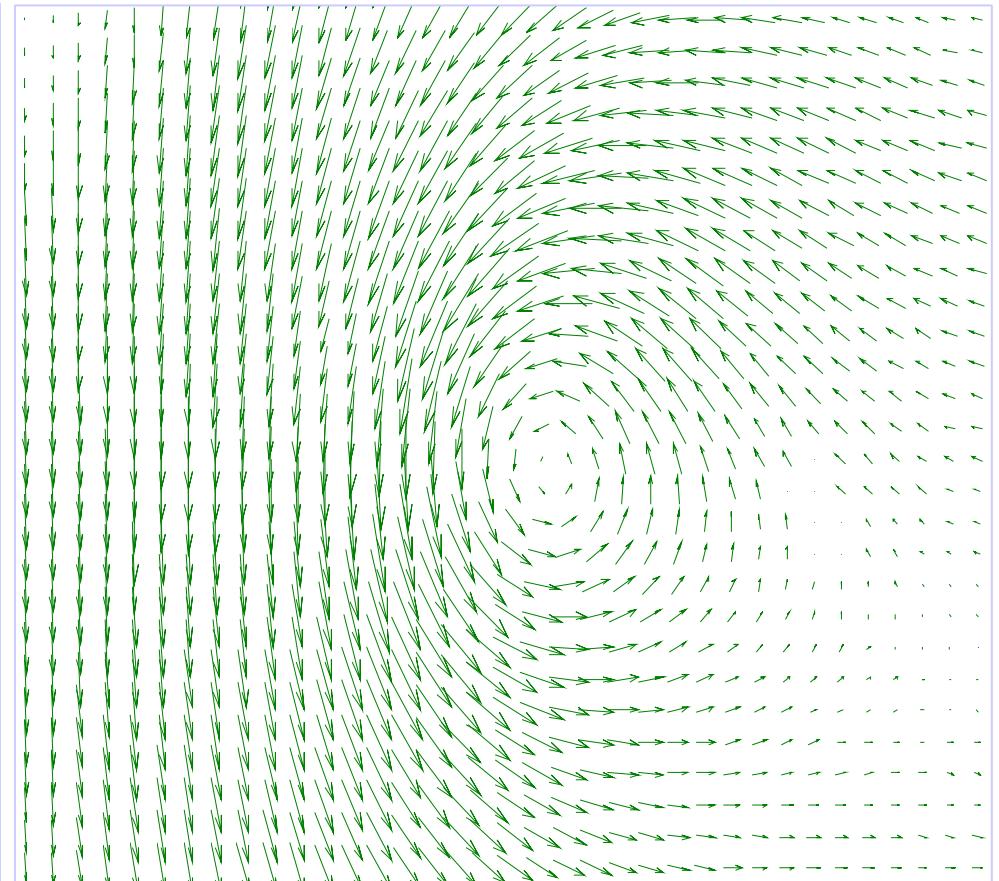
## Sample Results

$\sigma = -0.370$



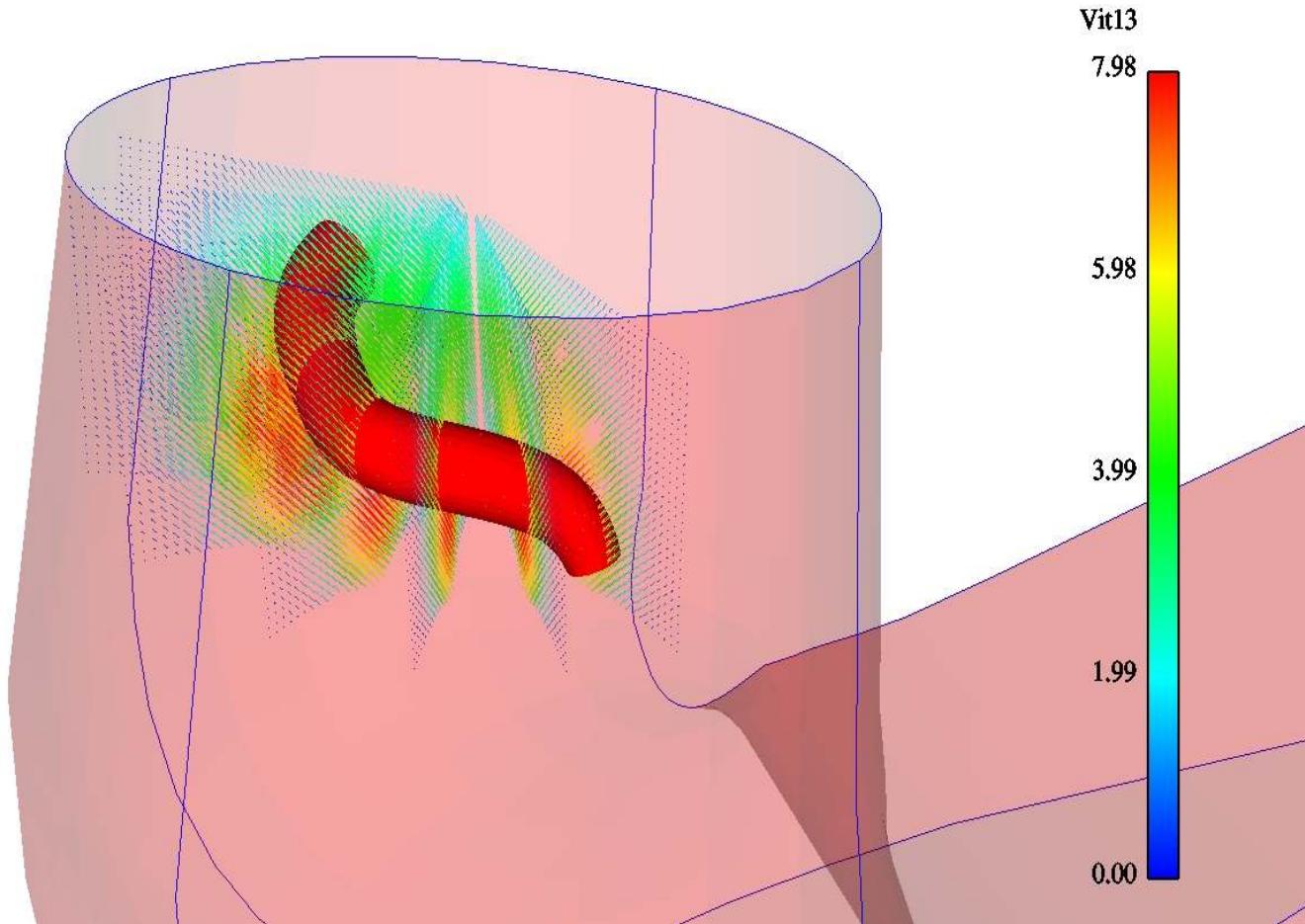
*Rope Configuration*

$\sigma = +0.427$



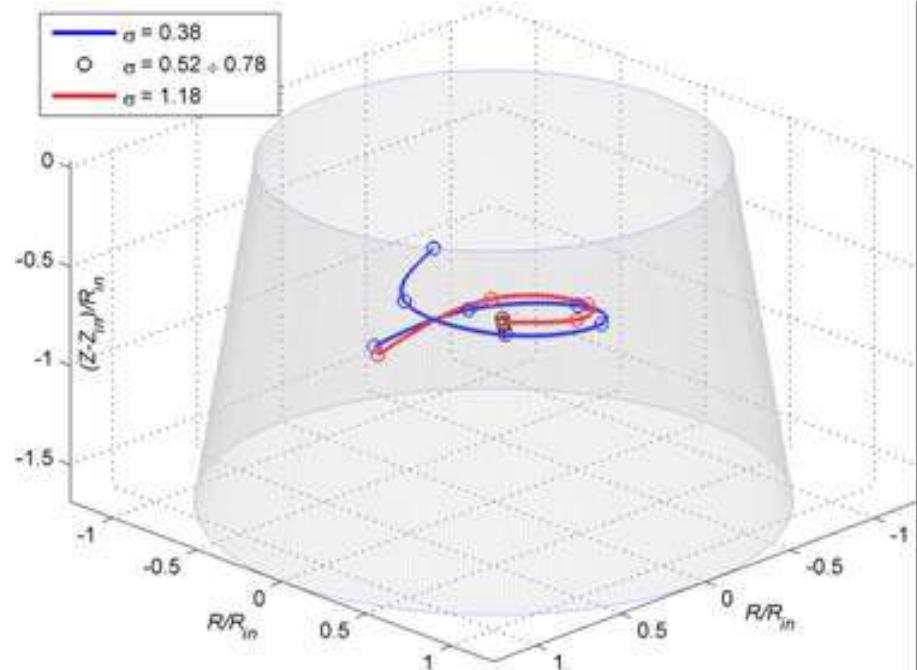
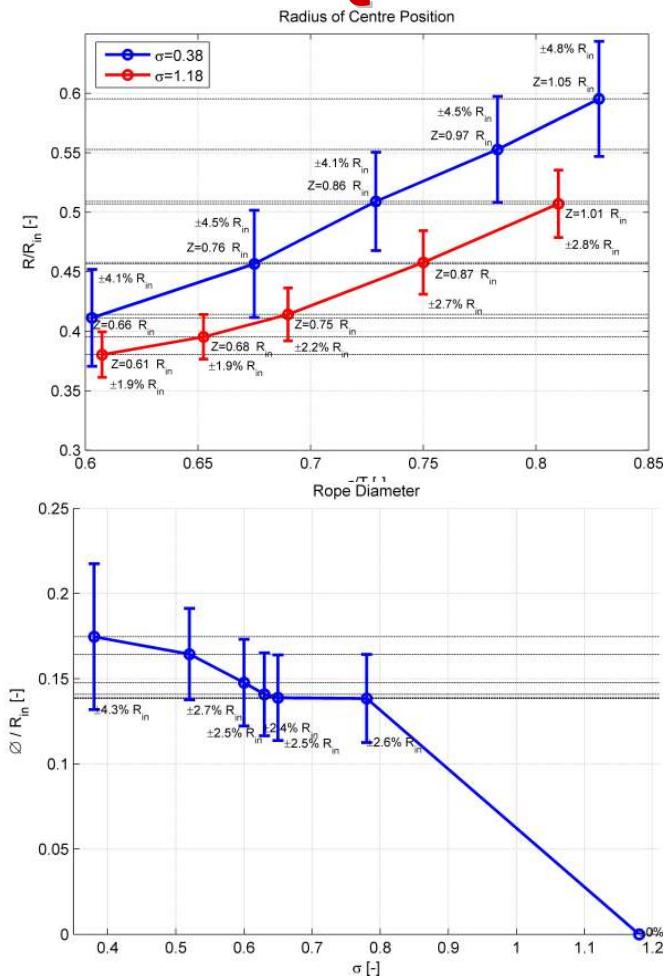
*Vortex Configuration*

## Phase Average Reconstruction

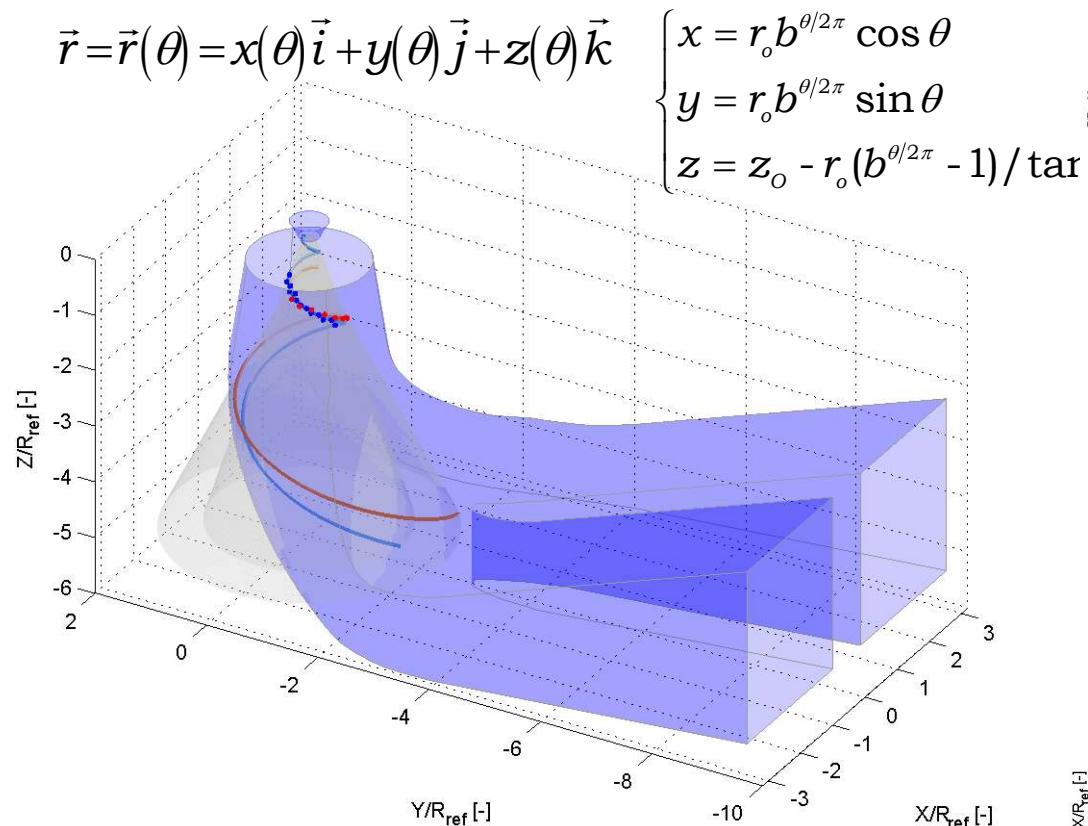


Iliescu M.S., Ciocan G.D., Avellan F.; "Two Phase PIV Measurements of a Partial Flow Rate Vortex Rope in a Francis Turbine"  
- proposed for Journal of Fluids Engineering, 2006

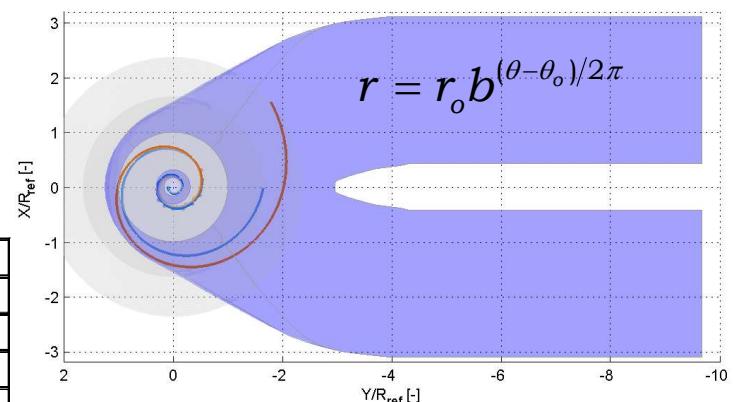
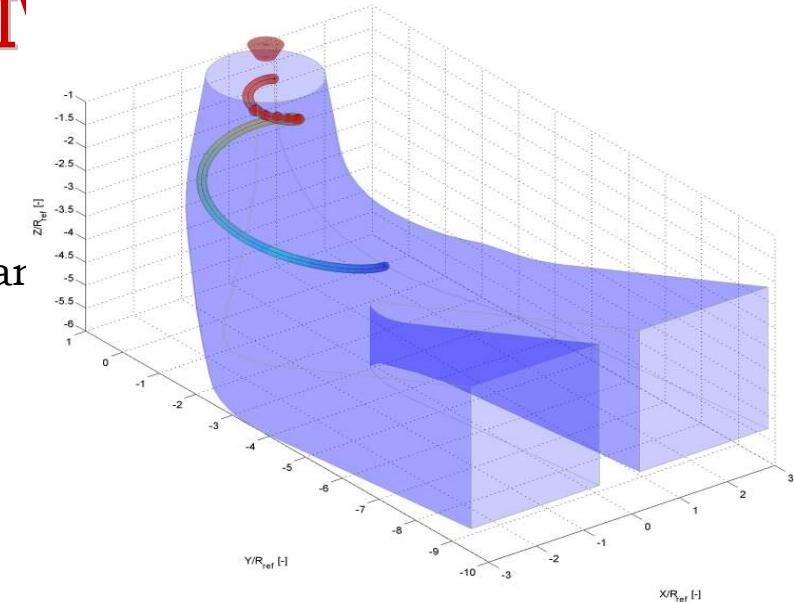
## Quantitative Rope Characterization



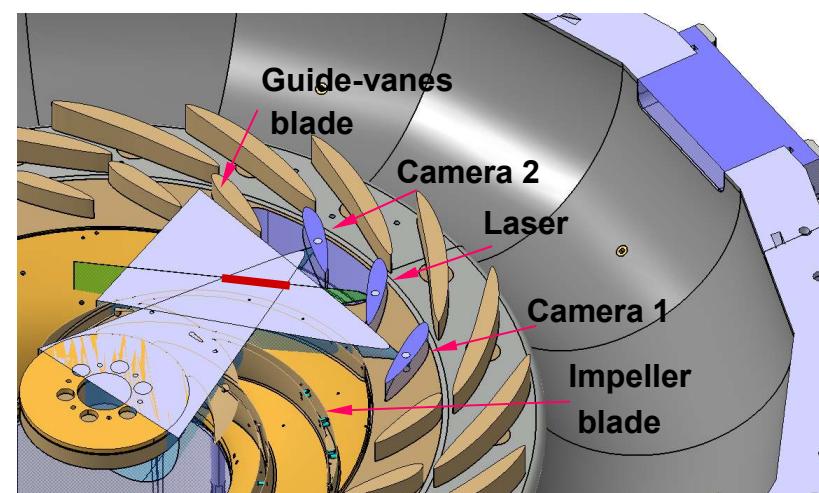
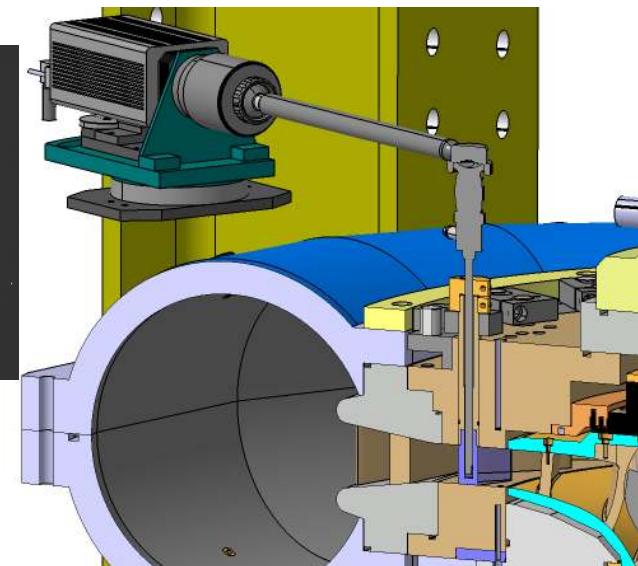
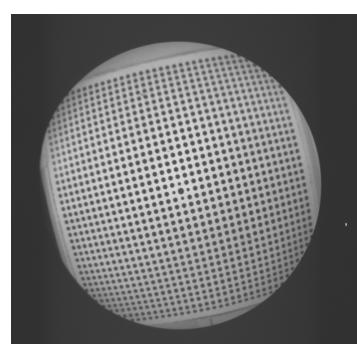
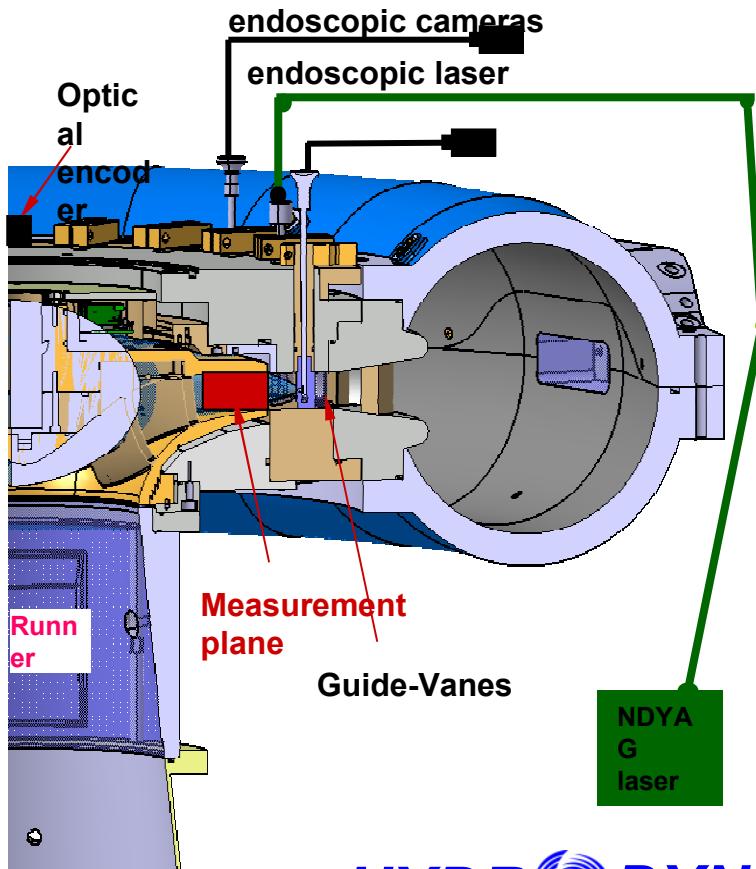
## 2 Phases PI



	Cavitation-free vortex	Vapors-core vortex
Initial radius	$r_o$	0.09
Initial depth	$z_o$	-0.615
Rate of radial growth	$b$	3.2
Cone angle	$\beta$	17°
		$r = r_o b^{(\theta-\theta_o)/2\pi}$



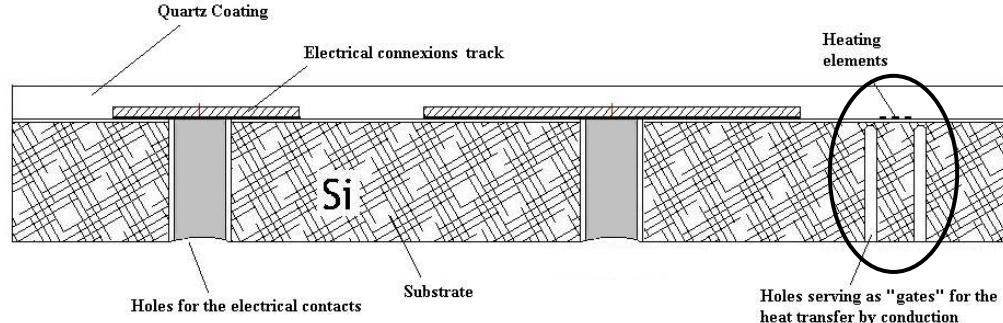
## Endoscopic PIV in the Runner



**HYDRO**DYNA

# **1. Design and Fabrication of the Miniature Probe**

## **SERIE I**



### **Validations:**

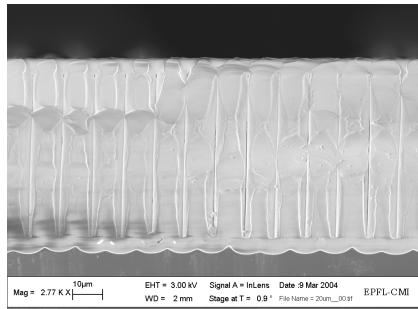
- Isolation between the surface of the probe, which comes in contact with the water, and the hot-film;
- There were chosen the materials for the hot film, the refilling holes.

### **Results:**

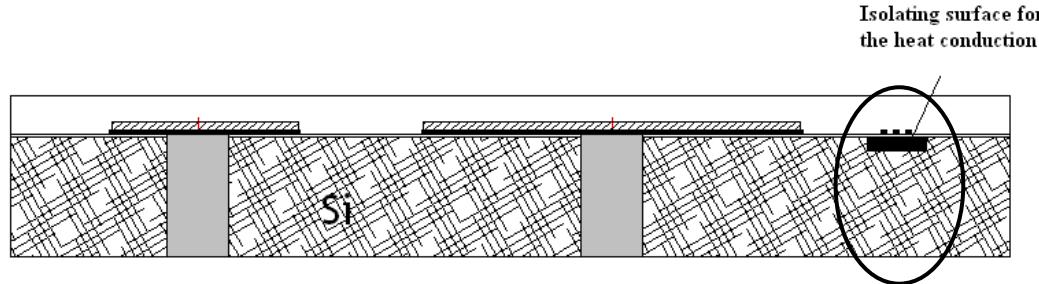
- a high thermal conduction, even with the gates;
- the detachment of the quartz on the electrical connexions;
- a very short " life time" in the water (~2-3 days), due to the high thermal conduction

# **1. Design and Fabrication of the Miniature Probe**

## SERIE II



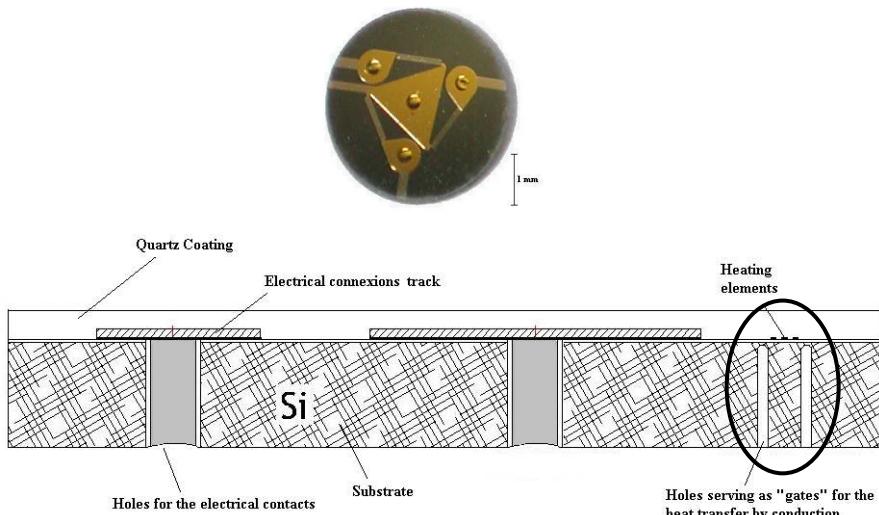
**Example of refilled Si pillars performed at CMI**



### **New studied parameters:**

- *The thickness of the silicon pillars, for optimizing the refill with the SiO<sub>2</sub> layer.*
- *The optimally forms and dimensions of the pillars in order to realise a compact and solid substrate*

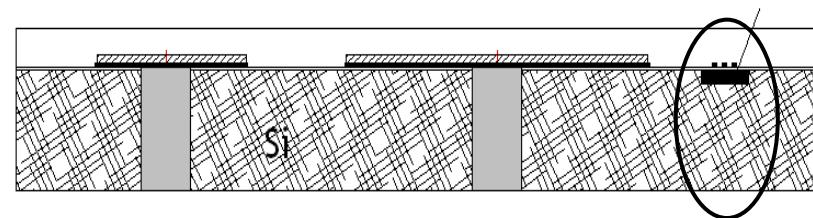
## 1. Design and Fabrication of the Miniature Probe



Serie I



Isolating surface for  
the heat conduction



Serie II (3  
wafers)

### Similarities

- ✓ Hot films, refilling holes, substrate, electrical connexions materials;
- ✓ Design

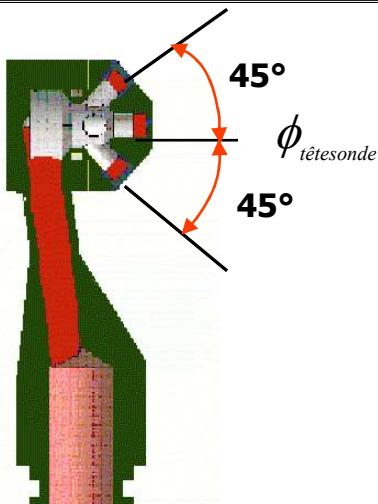
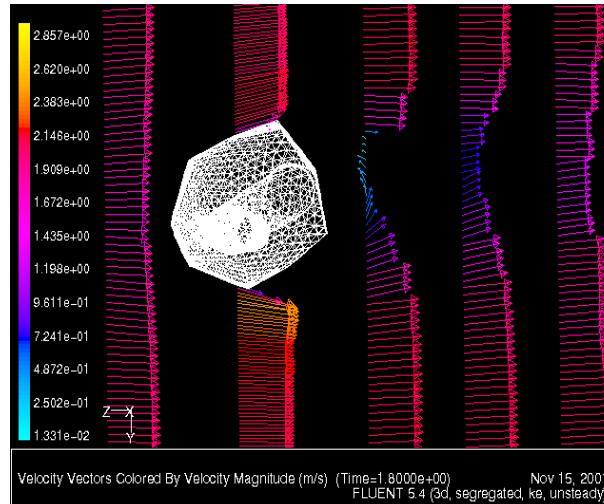
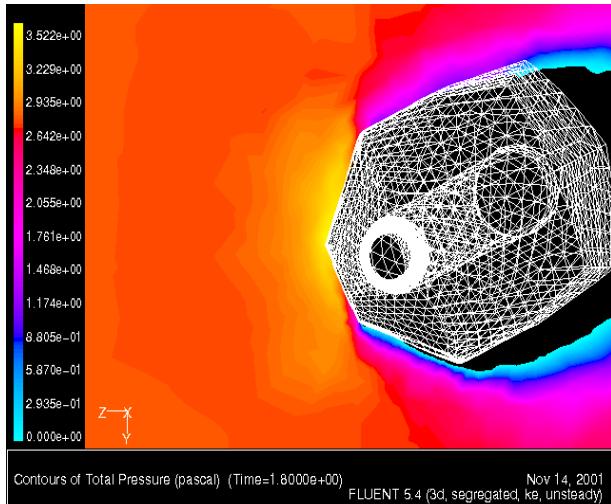
### Differences:

- ✓ Thermal isolation's realisation: gates against isolating surface
- ✓ New studied parameters:
  1. Thickness of the silicon pillars;
  2. Optimally forms and dimensions of the pillars

### Results:

- ✓ high thermal conduction;
- ✓ detachment of the quartz on the electrical connexions;
- ✓ very short "life time" in the water (~2-3 days).

## Unsteady Pressure Probes



- ✓ shape of the probe head = pyramid
- ✓ probe opening angle = 90°
- ✓ external diameter of the probe = 9.24 mm
- ✓ sensors: UNISENSOR 0-5 bar, D=2mm, H=1.5 mm

### Probe:

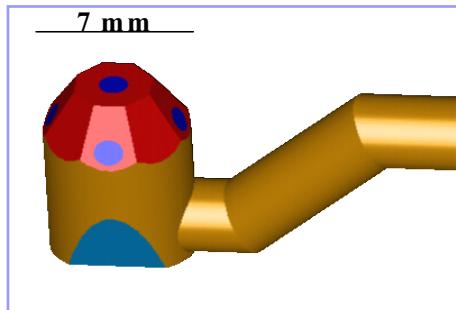
- ✓ angles 90° & 90°
- ✓ diameter 10 mm

1 calcul  $\alpha = 0^\circ, \tau = 0^\circ$

2 calcul  $\alpha = 25^\circ, \tau = 0^\circ$

## Unsteady Pressure Probes

Unsteady total pressure  
probe INPG



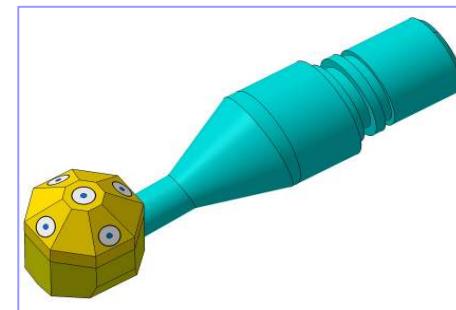
Angles :  $55^\circ$  &  $55^\circ$   
Sensitivity: 3 units F / $50^\circ\alpha$   
3 units G / $50^\circ\tau$

Unsteady total pressure  
probe LMH  
United Sensor Geometry



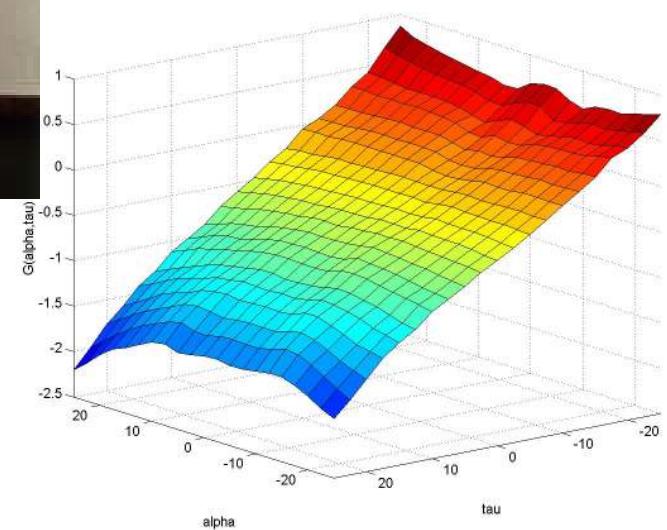
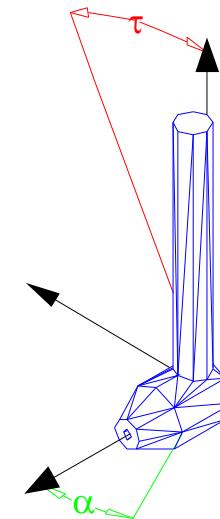
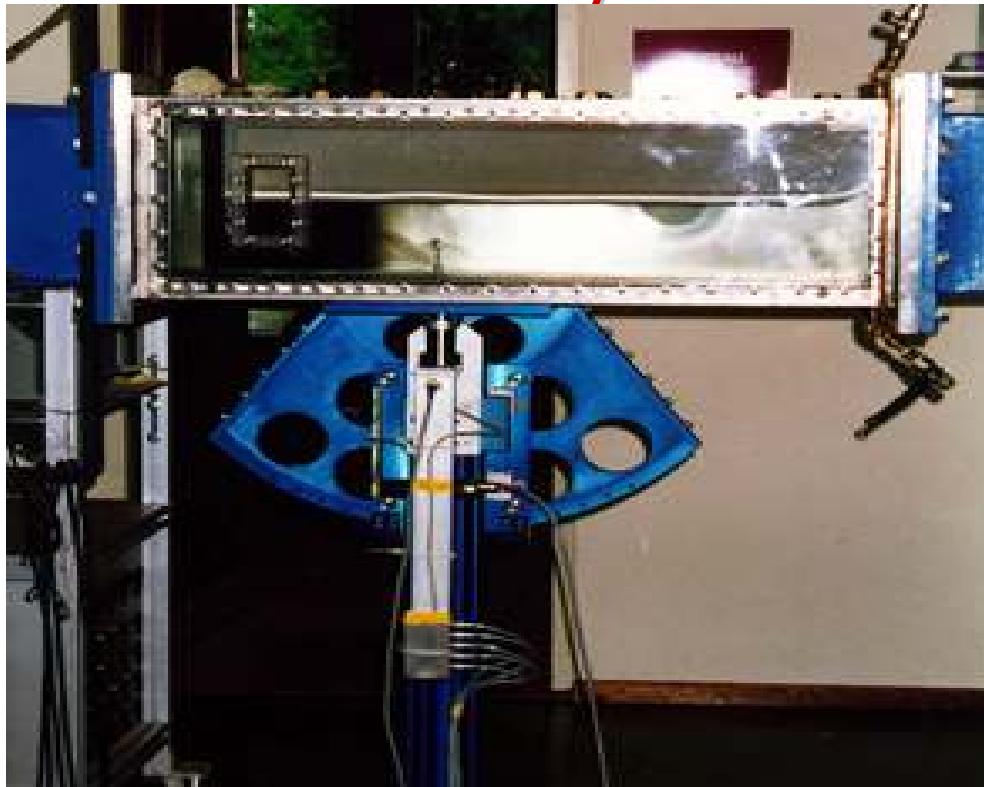
Angles:  $100^\circ$  &  $-90^\circ$   
Sensitivity: 8 units F / $50^\circ\alpha$   
1.4 units G / $50^\circ\tau$

Unsteady total pressure  
probe LMH



Angles:  $90^\circ$  &  $-90^\circ$   
Sensitivity: 4 units F / $50^\circ\alpha$   
4 units G / $50^\circ\tau$

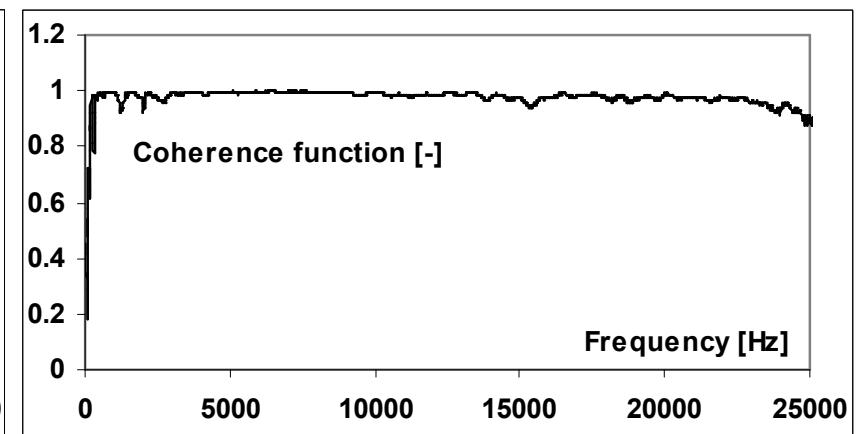
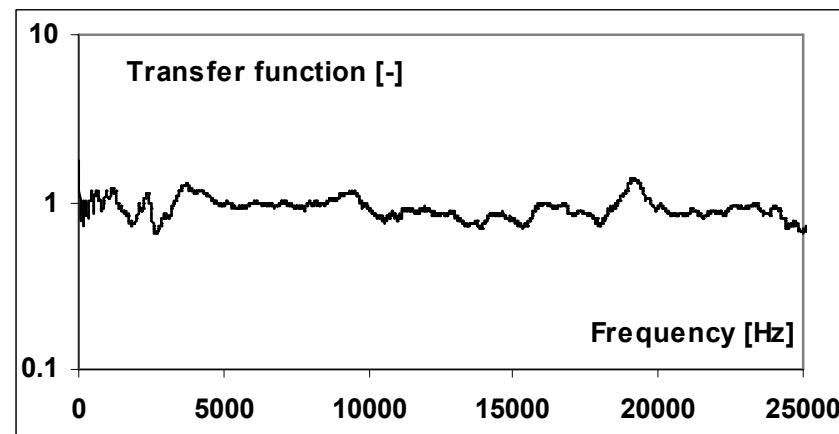
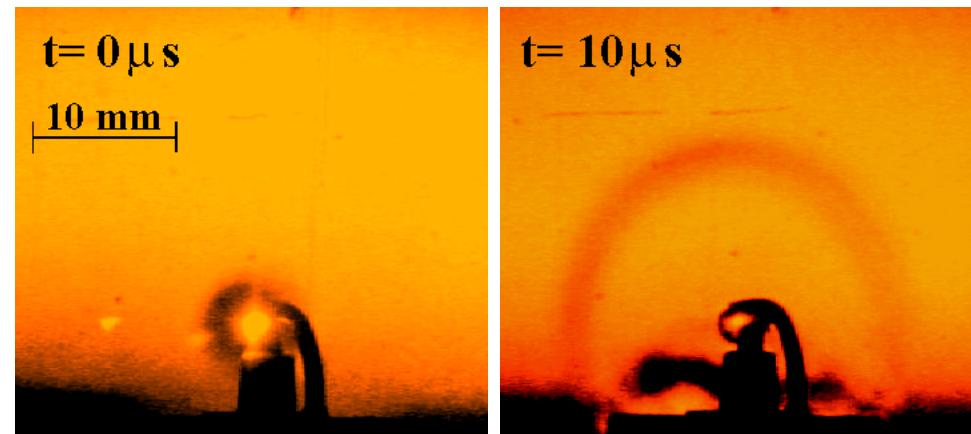
## Unsteady Pressure Probe



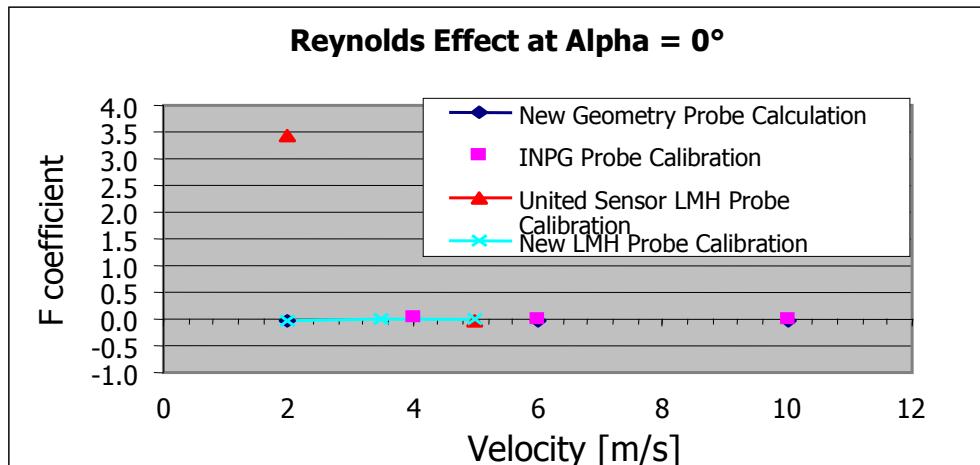
## Calibration

### Dynamic calibration

- Pressure wave obtained by bubble implosion
- Linear by 25 kHz

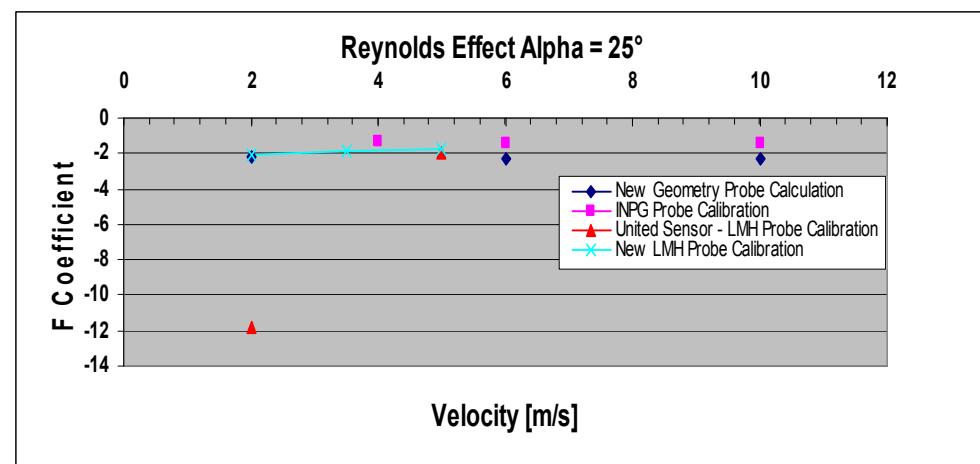


## Reynolds Effect



Probe numerical simulation :

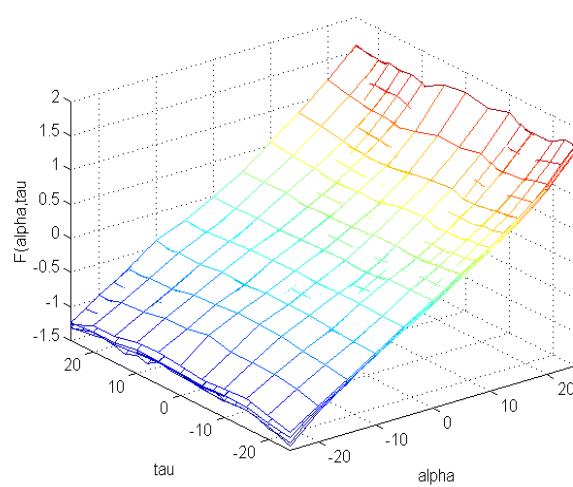
- ✓ *between 2 & 10 m/s*
- ✓ *variation F à 0° = 0%*
- ✓ *variation F à 25° less 4%*



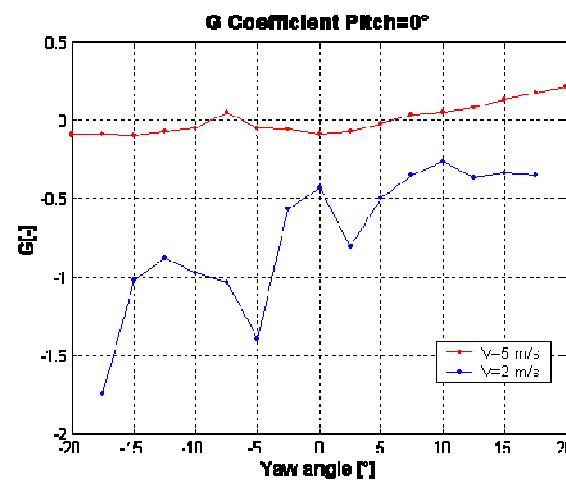
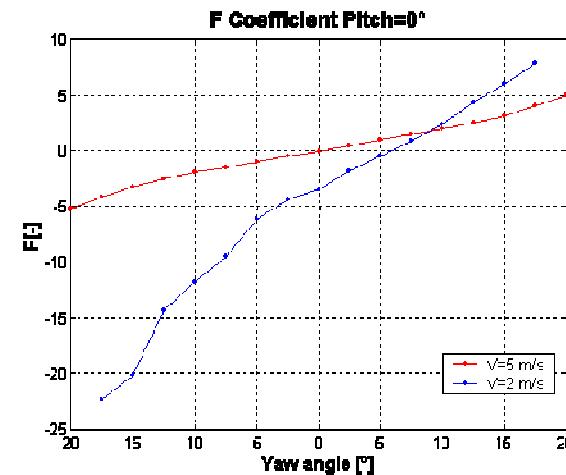
Probes calibration:

- ✓ *between 2 & 5 m/s*
- ✓ *variation F à 0° = 0%*
- ✓ *variation F à 25° less 2%*

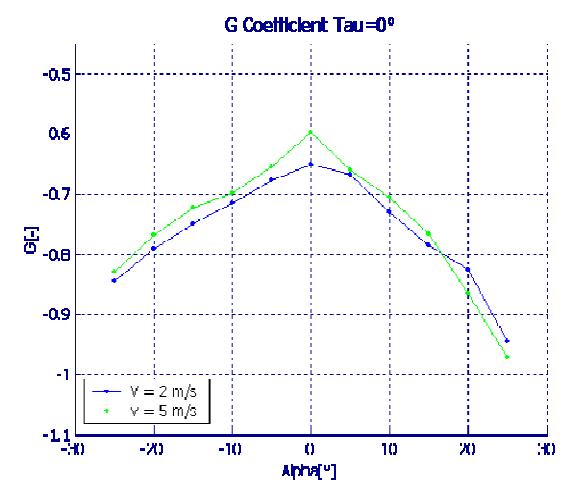
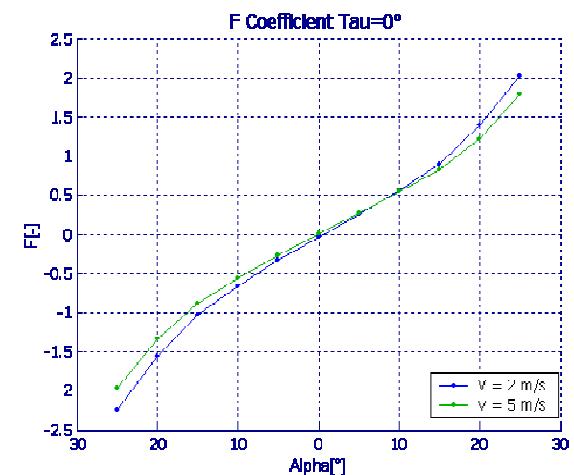
## Reynolds Effect



Sonde INPG 4 -10 m/s

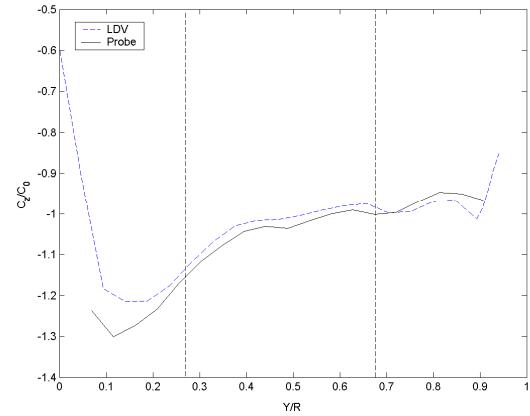
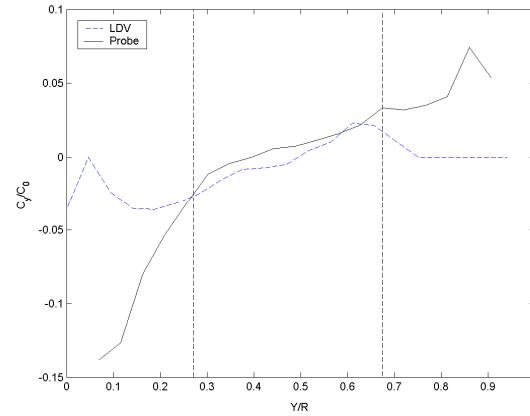
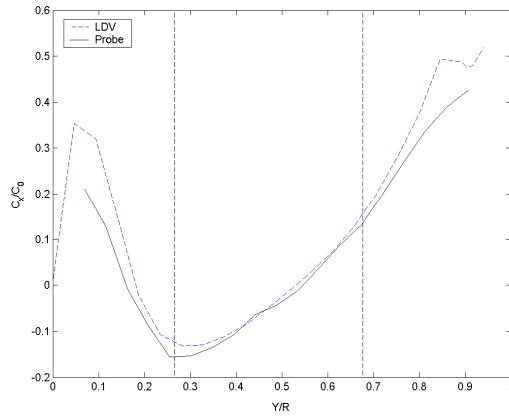


Sonde United Sensor - LMH 4 - 8 m/s



Sonde LMH 2 – (5) m/s

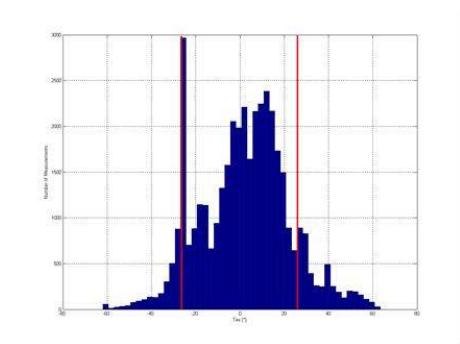
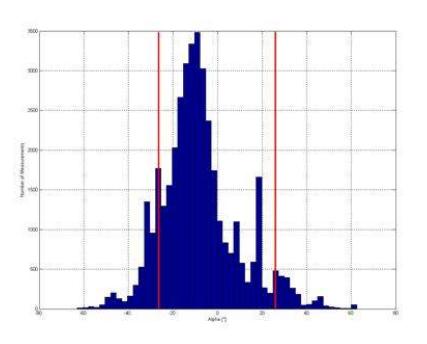
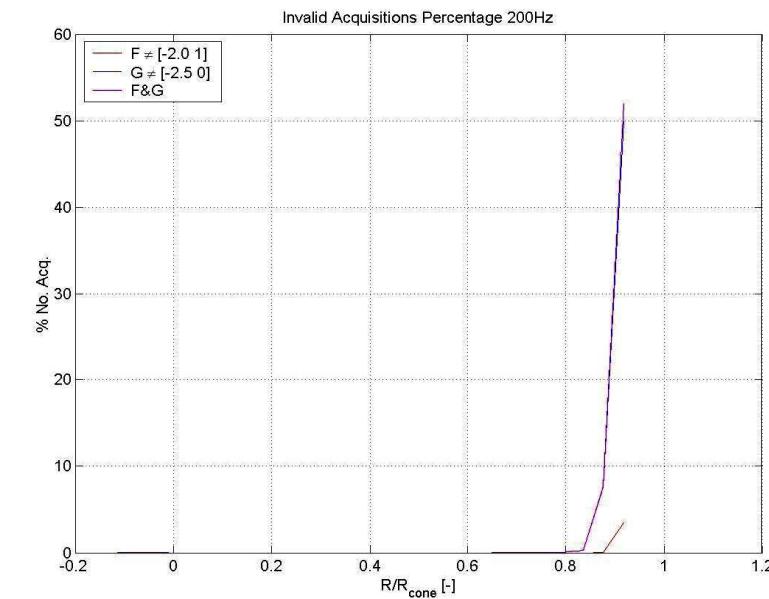
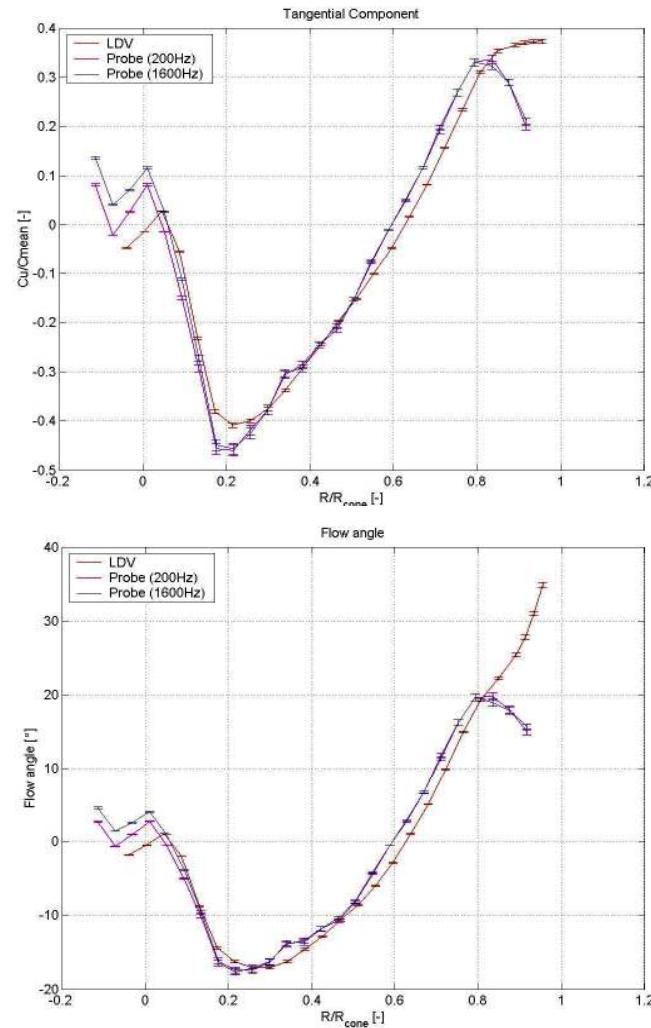
## Unsteady pressure measurements



✓ Validity domain of the probe accuracy

# Metode moderne pentru cercetarea in Masini Hidraulice

## Unsteady pressure measurements



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## Conclusion

- ✓ *Strong needs in the hydraulic machinery development:*
  - ✓ *New market needs*
  - ✓ *New developments with strong environmental, exploitation and operation constraints*
  
- ✓ *Can be insured by:*
  - ✓ *New high accuracy tools development: theoretical, experimental and numerical*
  - ✓ *Multidisciplinary approach to solve new topics*

